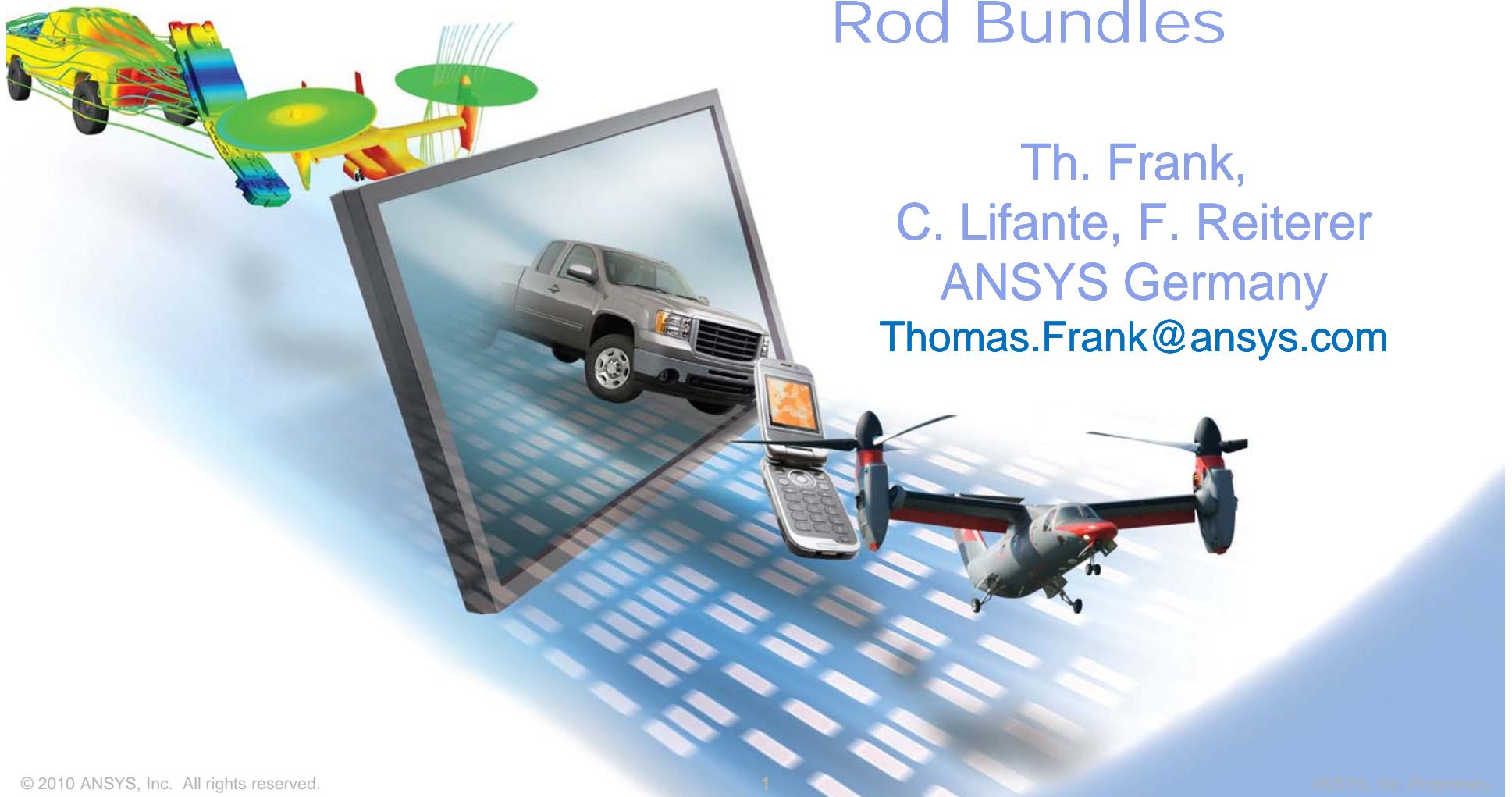




Advances in the Simulation of Boiling Steam-Water Flow through Fuel Assembly Subchannels and Rod Bundles

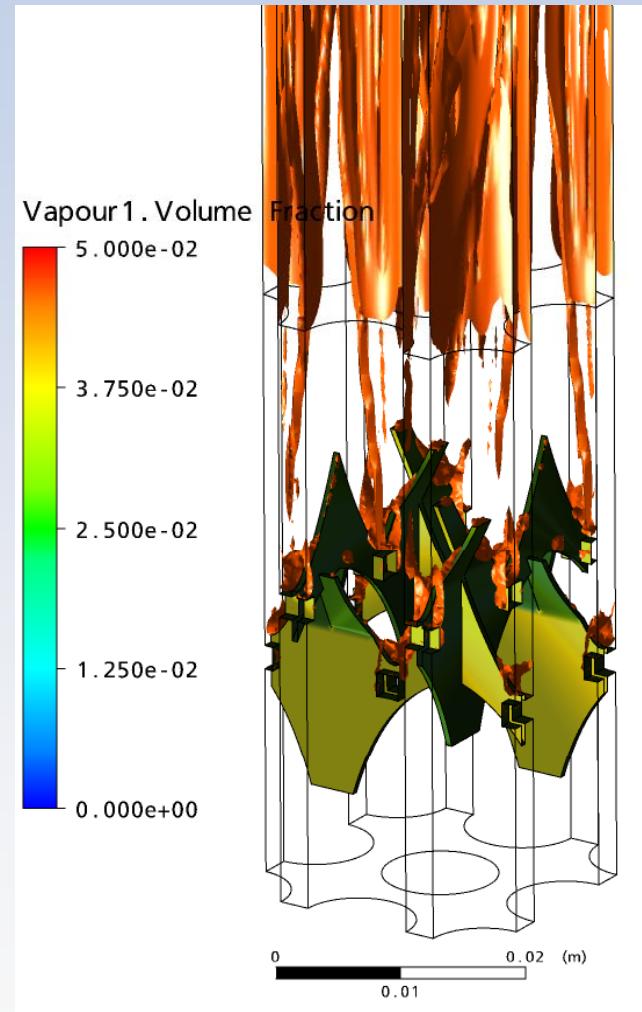
Th. Frank,
C. Lifante, F. Reiterer
ANSYS Germany
Thomas.Frank@ansys.com



Outline



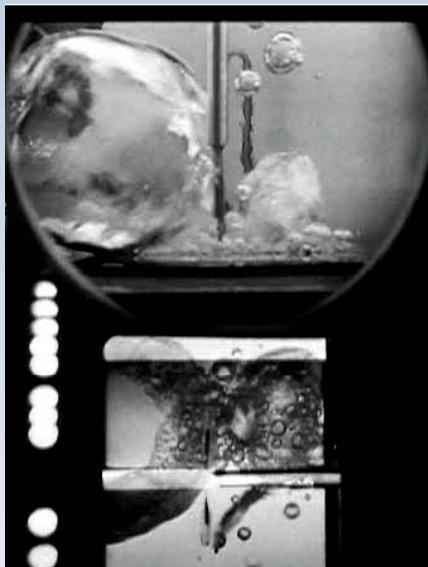
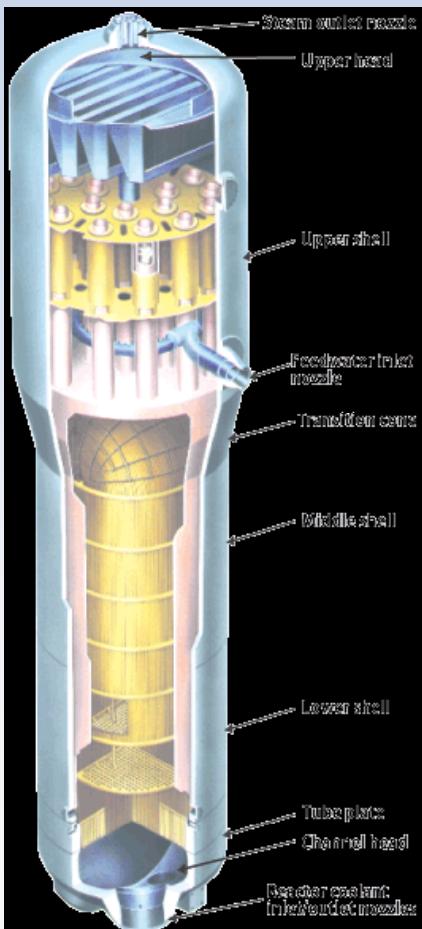
- Introduction
- Development of subcooled nucleate boiling model:
 - The modified RPI wall boiling model
 - Extensions to the RPI model
 - Coupling of RPI & MUSIG
- Validation & application of the boiling model in ANSYS CFD
 - Boiling & recondensation
 - FRIGG loop: Boiling in heated rod bundles
- Summary & Outlook



Boiling Flow Applications

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Steam Generators

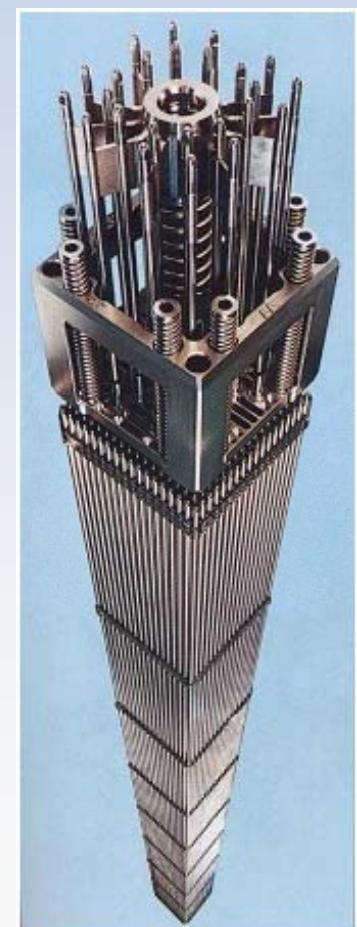


Process Technology

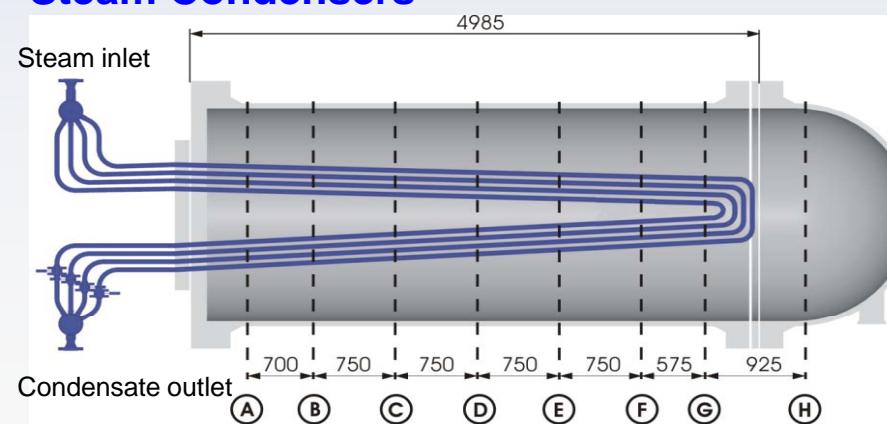


Engine cooling water jackets

Fuel Assemblies



Steam Condensers



Wall Boiling Modeling

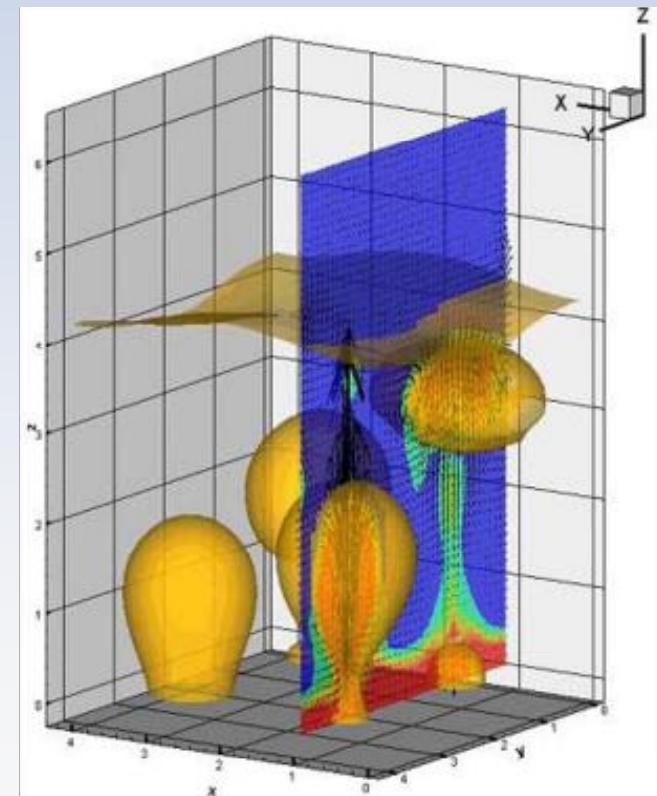


Why special modeling for wall boiling?

- For subcooled flows with superheated walls, standard thermal phase change models for bulk boiling/condensation will underpredict mass transfer rates
- Accounts for steam bubble growth on nucleation sites and bubble departure
- Mechanistic model for wall driven boiling

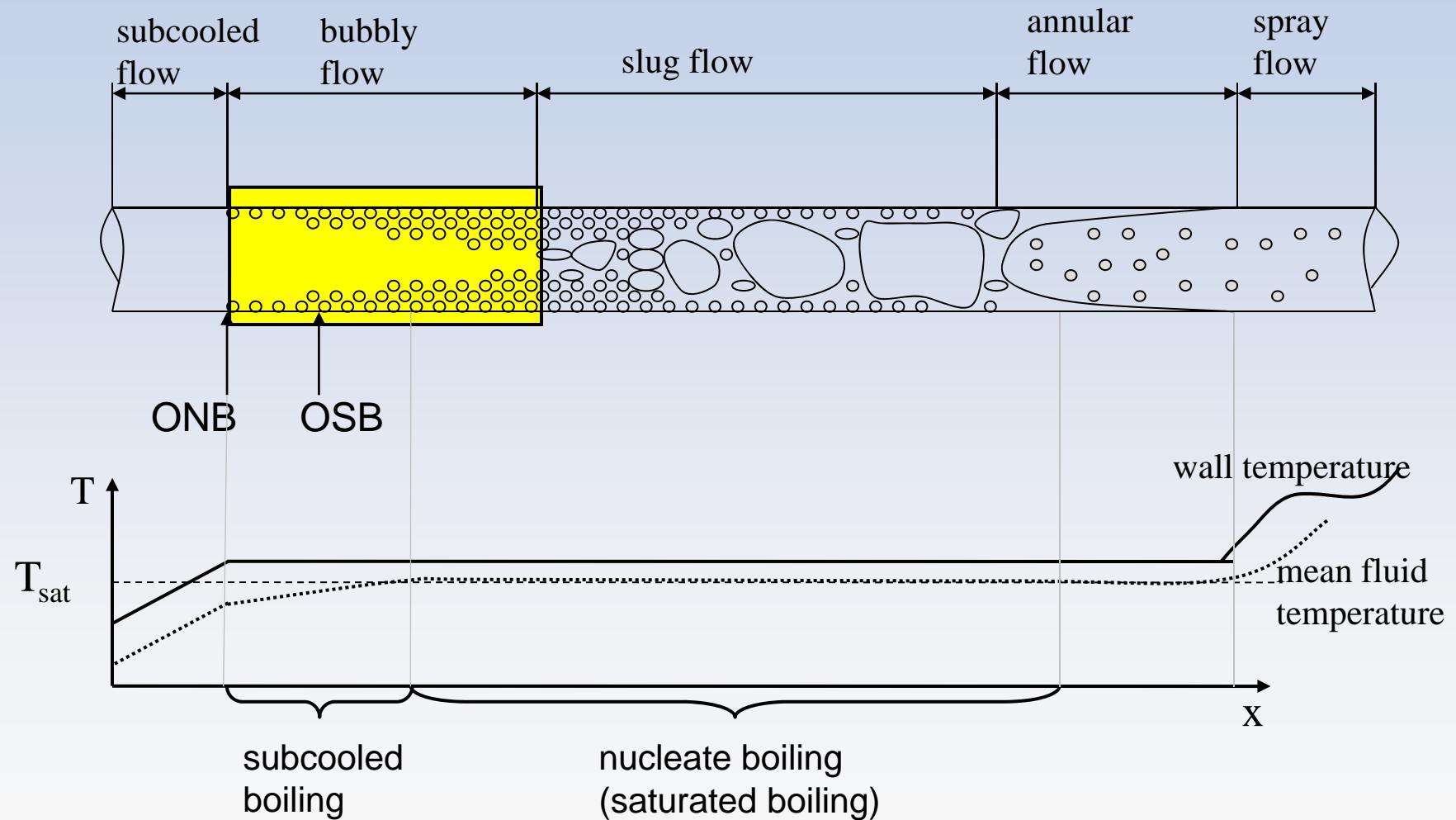
Model outline:

- Mechanistic wall heat flux splitting
→ convective heat transfer,
evaporation, quenching
- Empirical submodels required for closure
- Available for different BC's:
prescribed T_{wall} or q_{wall} , CHT walls
- Activated per boundary patch with individual T_{wall} or q_{wall}



Multiphase Flow Regimes for Boiling Water Flow

ANSYS®



Mechanistic wall heat partitioning model:

$$\dot{q}_{Wall} = \dot{q}_F + \dot{q}_Q + \dot{q}_E$$

convective heat flux

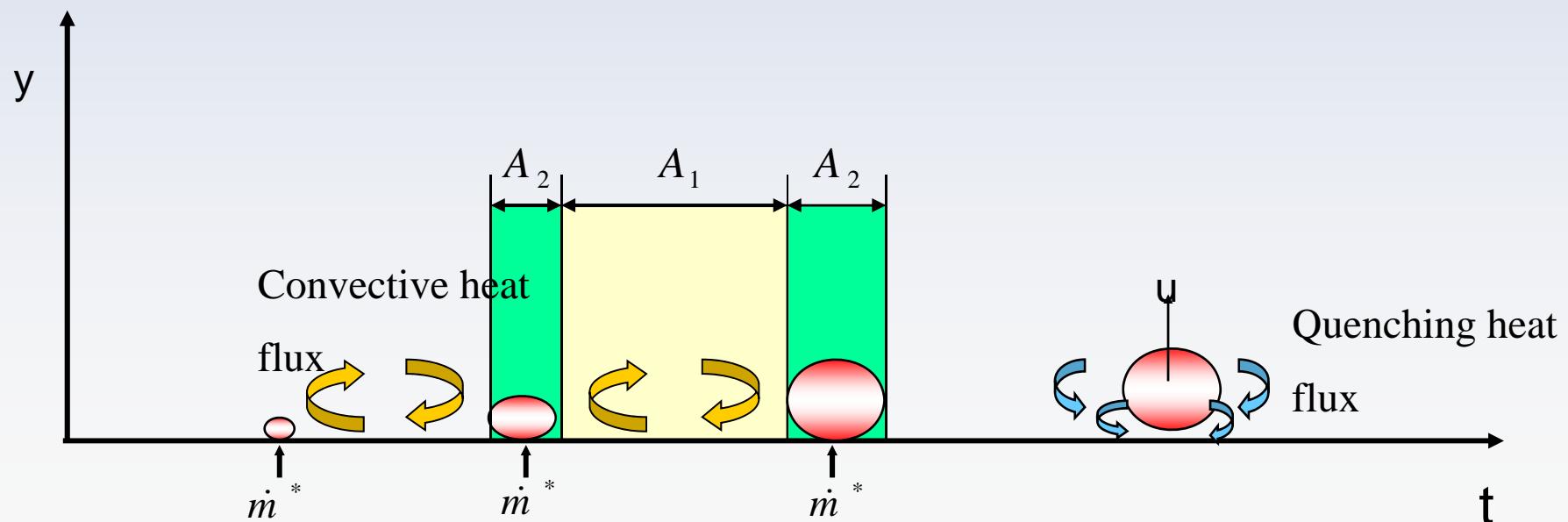
$$\dot{q}_F = A_1 \cdot h_F \cdot (T_W - T_L)$$

quenching heat flux

$$\dot{q}_Q = A_2 \cdot h_Q \cdot (T_W - T_L)$$

evaporation heat flux

$$\dot{q}_E = \dot{m} \cdot (h_G - h_L)$$



RPI-Wall Boiling Model – Submodels for Model Closure



Submodels for closure of RPI wall boiling model:

- Nucleation site density: Lemmert & Chawla , User Defined
- Bubble departure diameter:
Tolubinski & Kostanchuk, Unal, Fritz, User Defined
- Bubble detachment frequency:
Terminal rise velocity over Departure Diameter, User Defined
- Bubble waiting time:
Proportional to Detachment Period, User Defined
- Quenching heat transfer: Del Valle & Kenning, User Defined
- Turbulent Wall Function for liquid convective heat transfer coefficient
- Correlation for bulk flow mean bubble diameter required:
→ e.g. Kurul & Podowski correlation via CCL
- Supported combination of wall boiling & CHT in the solid
 - GGI & 1:1 solid-fluid interfaces

RPI Wall Boiling Model in the ANSYS CFX-Pre 12.0 GUI



Basic Settings | Fluid Models | Fluid Specific Models | **Fluid Pair Models | Initialisation |**

Mass Transfer

Option: Phase Change (highlighted with red circle)

Phase Change Model

Option: Thermal Phase Change (highlighted with red circle)

Saturation Temperature

Saturation Temp.: SaturTemper (highlighted with red circle)

Wall Boiling Model

Option: RPI Model (highlighted with red circle)

Fixed γ_{plus} for Liquid Subcooling

Mass Source Under Relaxation

Bubble Departure Diameter

Wall Nucleation Site Density

Bubble Detachment Frequency

Bubble Waiting Time

Liquid Quenching Heat Transfer Coefficient

Bubble Diam. Influence Factor

Max. Area Frac. of Bubble Influence

Heat Transfer

Option: Two Resistance (highlighted with red circle)

Liquid Heat Transfer

Option: Ranz Marshall (highlighted with red circle)

Vapour Heat Transfer

Option: Zero Resistance (highlighted with red circle)

Wall Boiling Model

Option: RPI Model

Fixed γ_{plus} for Liquid Subcooling

Fixed γ_{plus} : 250.0

Mass Source Under Relaxation

Mass Source Under Rel: 0.1

Bubble Departure Diameter

Option: Tolubinski Kostanchuk

Ref. Departure Diam.: 0.6E-3 [m]

Max. Departure Diam.: 1.4E-3 [m]

Liquid Subcooling Scale: 45.0 [K]

Wall Nucleation Site Density

Option: Lemmert Chawla

Site Density: 7.9384e5 [m⁻²]

Ref. Wall Superheat: 10.0 [K]

Power Law Index: 1.805

Bubble Detachment Frequency

Option: Terminal Velocity over Departure Diameter

Drag Coefficient: 1.0

Bubble Waiting Time

Option: Proportional to Detachment Period

Waiting Time Fraction: 0.8

Liquid Quenching Heat Transfer Coefficient

Option: Del Valle Kenning

Bubble Diam. Influence Factor

Factor: 2.0

Max. Area Frac. of Bubble Influence

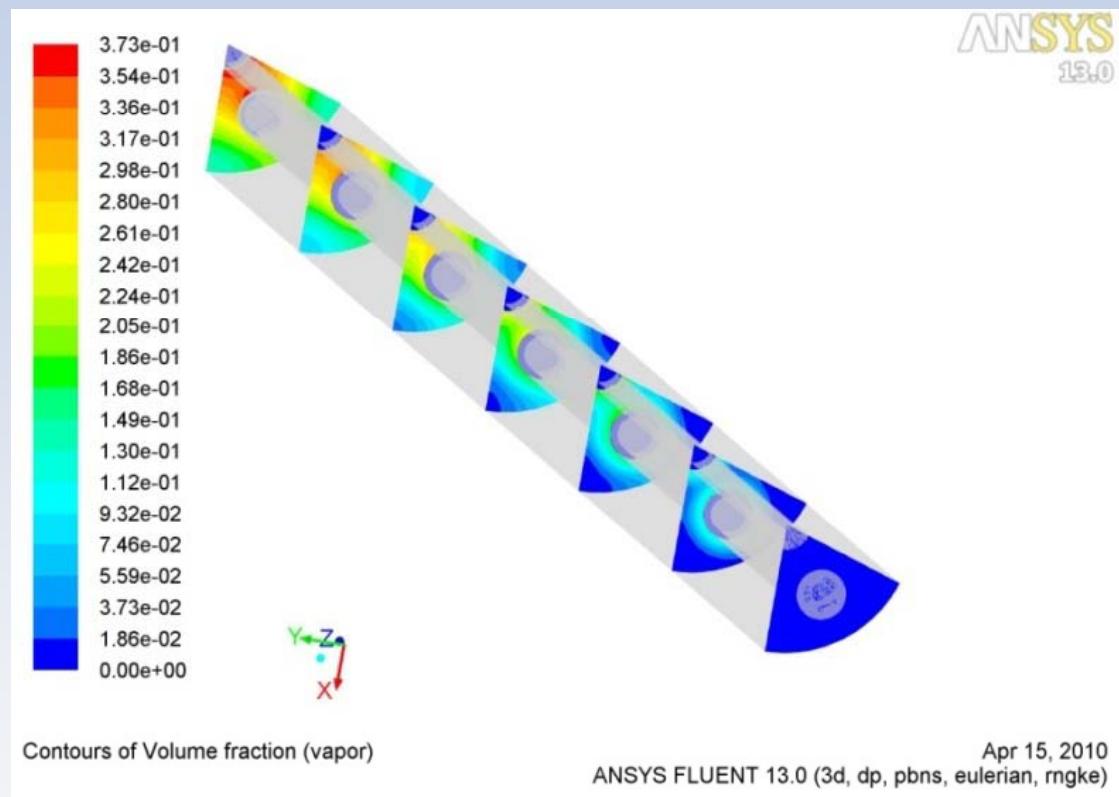
Max. Area Fraction: 0.5

Buttons: OK (highlighted with red circle), Apply, Close

ANSYS Fluent 13.0 Wall Boiling Modeling



- ANSYS Fluent 13.0:
 - Based on same RPI nucleate boiling & heat flux partitioning model
 - Non-equilibrium subcooled boiling
 - Supports super-heated vapor (convective heat flux to vapor)



**Contours of vapor volume fraction
in a heated rod bundle**

ANSYS CFX R&D Development Work in Progress



- Ongoing R&D and development:
 - Provide more user interfaces to the RPI boiling model
 - User defined area fractions A_1 and A_2
 - User defined terms for convective, quenching and evaporative heat fluxes Q_F , Q_Q , Q_E
 - User defined **4th component of wall heat partitioning**,
e.g. convective heat flux to vapor
 - CFX5Pre GUI extension
 - Extended output to CFD-Post
 - **Coupling of RPI wall boiling & MUSIG**
- All extensions are part of a collaborative R&D project with FZD → customized CFX solver

New Capabilities: CCL Access to Area Fractions

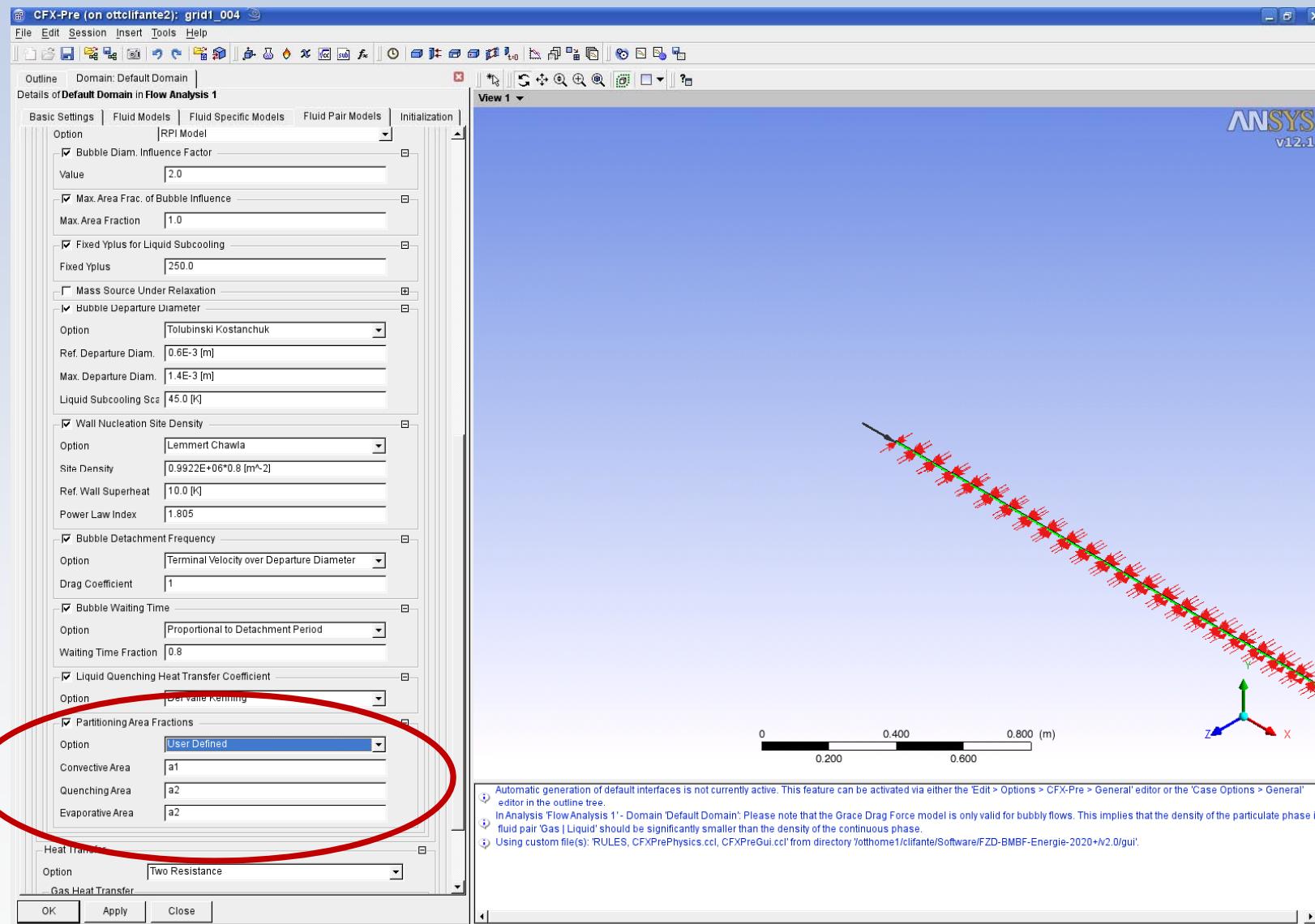


```
MASS TRANSFER:  
  Option = Phase Change  
PHASE CHANGE MODEL:  
  Option = Thermal Phase Change  
WALL BOILING MODEL:  
  Bubble Diameter Influence Factor = 2.0  
  Fixed Yplus for Liquid Subcooling = 250.0  
  Maximum Area Fraction of Bubble Influence = 1.0  
  Option = RPI Model  
BUBBLE DEPARTURE DIAMETER:  
  Liquid Subcooling Scale = 45.0 [K]  
  Maximum Departure Diameter = 1.4E-3 [m]  
  Option = Tolubinski Kostanchuk  
  Reference Departure Diameter = 0.6E-3 [m]  
END  
BUBBLE DETACHMENT FREQUENCY:  
  Drag Coefficient = 1  
  Option = Terminal Velocity over Departure Diameter  
END  
BUBBLE WAITING TIME:  
  Option = Proportional to Detachment Period  
  Waiting Time Fraction = 0.8  
END  
LIQUID QUENCHING HEAT TRANSFER COEFFICIENT:  
  Option = Del Valle Kenning  
END  
PARTITIONING AREA FRACTIONS:  
  Convective Area = a1  
  Evaporative Area = a2  
  Option = User Defined  
  Quenching Area = a2  
END  
WALL NUCLEATION SITE DENSITY:  
  Option = Lemmert Chawla  
  Power Law Index = 1.805  
  Reference Nucleation Site Density = 0.9922E+06*0.8 [m^-2]  
  Reference Wall Superheat = 10.0 [K]  
END  
END  
END
```

- **WALL BOILING MODEL**
 - **PARTITIONING AREA FRACTIONS**
 - **Option = Standard / User Defined**
 - **Under User Defined *convective, quenching* and *evaporative* area can be introduced**

New capabilities: CFX5Pre GUI Extension

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CCL & User Routine for 4th Wall Heat Partitioning Component



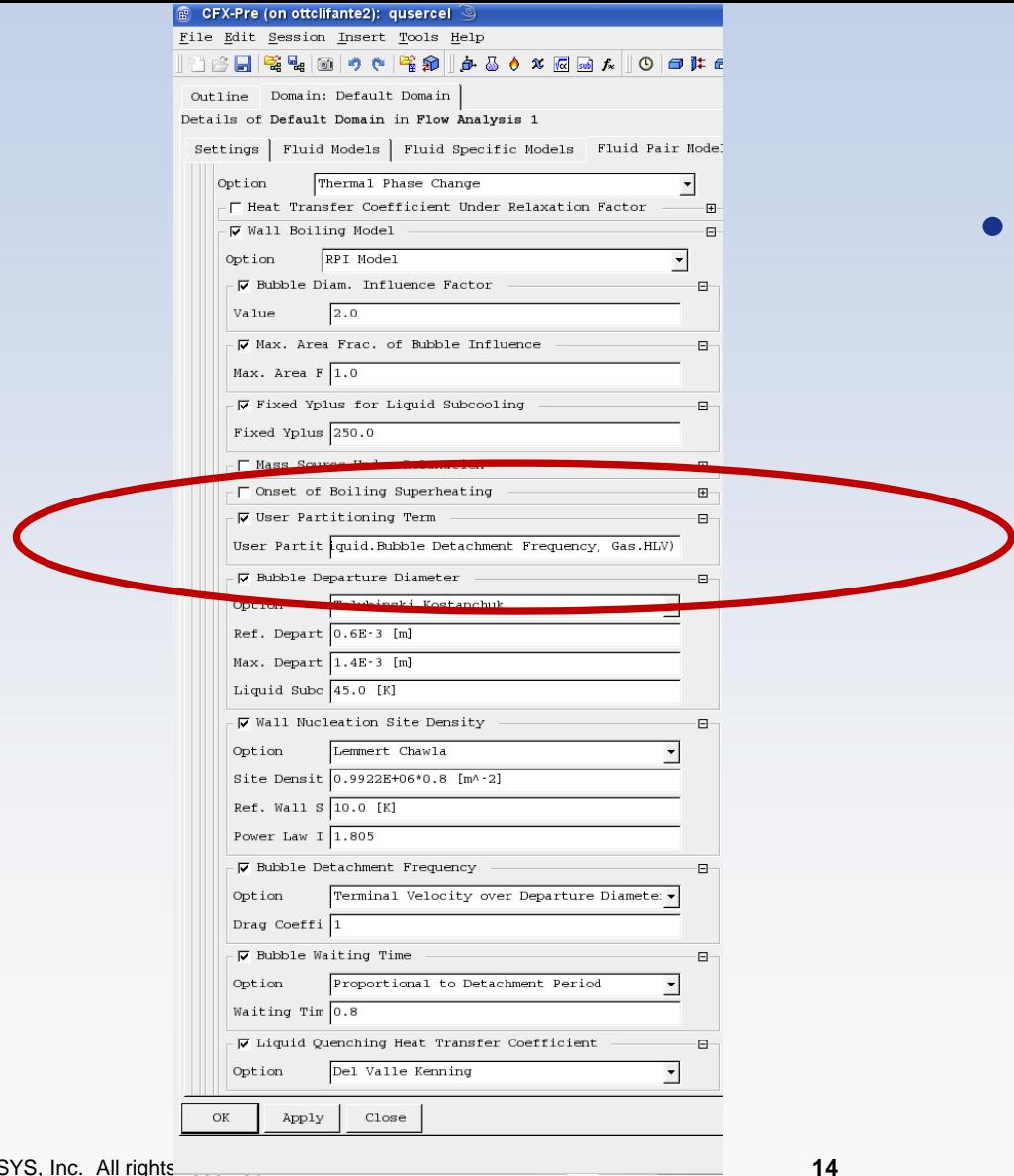
- **Introduction of 4th component of the wall heat flux partitioning via CCL or User Fortran**

```
Option = Fluid Dependent
END
FLUID PAIR: Gas | Liquid
INTERPHASE HEAT TRANSFER:
    Option = Two Resistance
        FLUID1 INTERPHASE HEAT TRANSFER:
            Option = Zero Resistance
        END
        FLUID2 INTERPHASE HEAT TRANSFER:
            Option = Ranz Marshall
        END
    END
INTERPHASE TRANSFER MODEL:
    Interfacial Area Density = AreaDensity
    Maximum Volume Fraction for Area Density = MaxVFForArea
    Minimum Volume Fraction for Area Density = MinVFForArea
    Option = Particle Model
END
MASS TRANSFER:
    Option = Phase Change
PHASE CHANGE MODEL:
    Option = Thermal Phase Change
WALL BOILING MODEL:
    Bubble Diameter Influence Factor = 2.0
    Fixed Yplus for Liquid Subcooling = 250.0
    Maximum Area Fraction of Bubble Influence = 1.0
    Option = RPI Model
User Partitioning Term = USERPARTTERM(Gas | Liquid.Bubble \
    Departure Diameter,Gas | Liquid.Nucleation Site Density, Gas | \
    Liquid.Temperature Superheating, Gas | Liquid.Temperature \
    Subcooling, Gas.Density,Gas | Liquid.Bubble Detachment \
    Frequency, Gas.HLV)
BUBBLE DEPARTURE DIAMETER:
    Liquid Subcooling Scale = 45.0 [K]
    Maximum Departure Diameter = 1.4E-3 [m]
    Option = Tolubinski Kostanchuk
    Reference Departure Diameter = 0.6E-3 [m]
END
BUBBLE DETACHMENT FREQUENCY:
    Drag Coefficient = 1
    Option = Terminal Velocity over Departure Diameter
END
BUBBLE WAITING TIME:
    Option = Proportional to Detachment Period
    Waiting Time Fraction = 0.8
END
LIQUID QUENCHING HEAT TRANSFER COEFFICIENT:
    Option = Del Valle Kenning
END
PARTITIONING AREA FRACTIONS:
    Convective Area = al
    Evaporative Area = 0.0
    Option = User Defined
    Quenching Area = a2
END
```

CCL & User Routine for 4th Wall Heat Partitioning Component

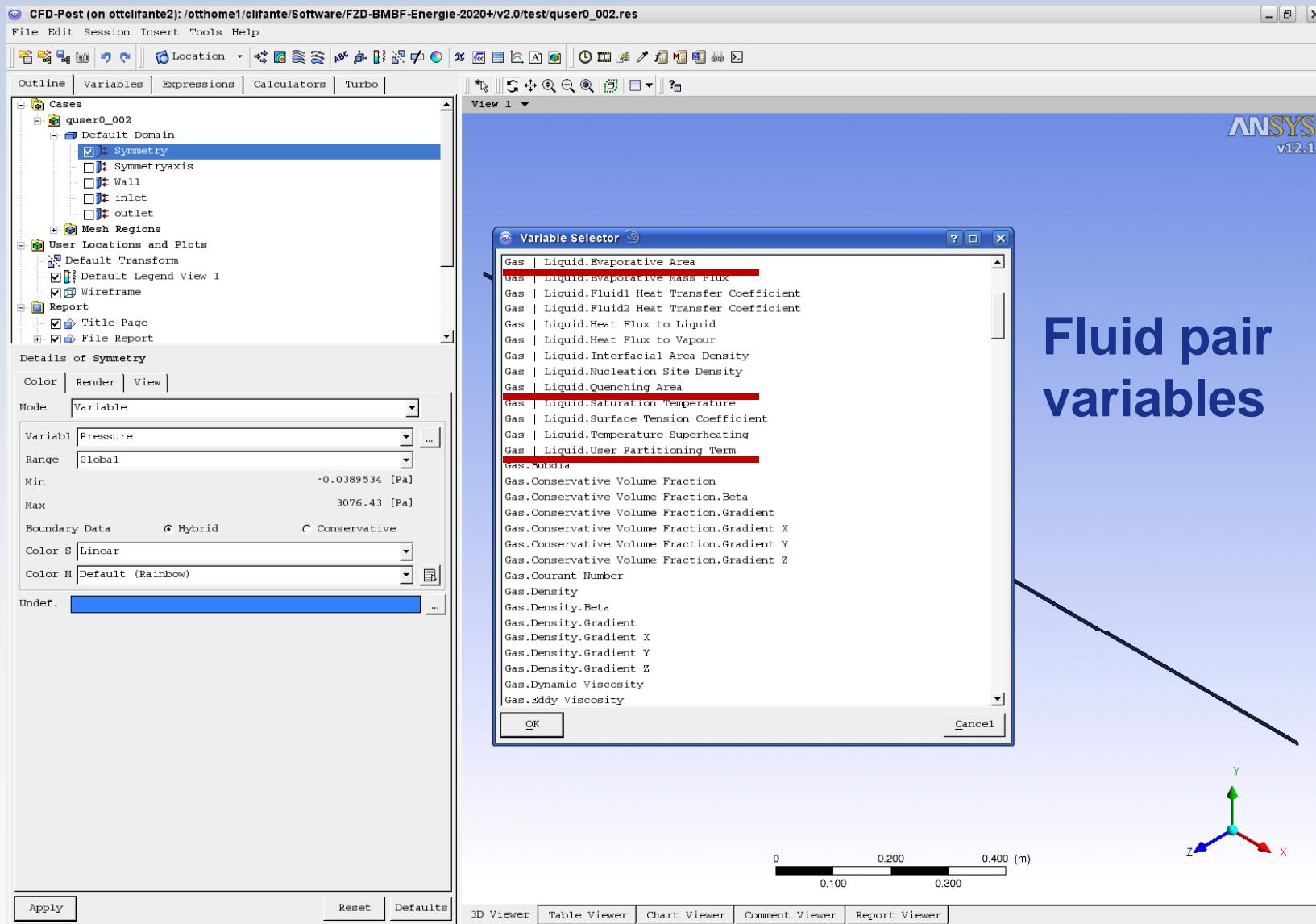


- Customization of CFX5Pre for the extension of the RPI wall heat flux partitioning algorithm
→ 4th component of the wall heat flux splitting



Extended CFX5Post Output

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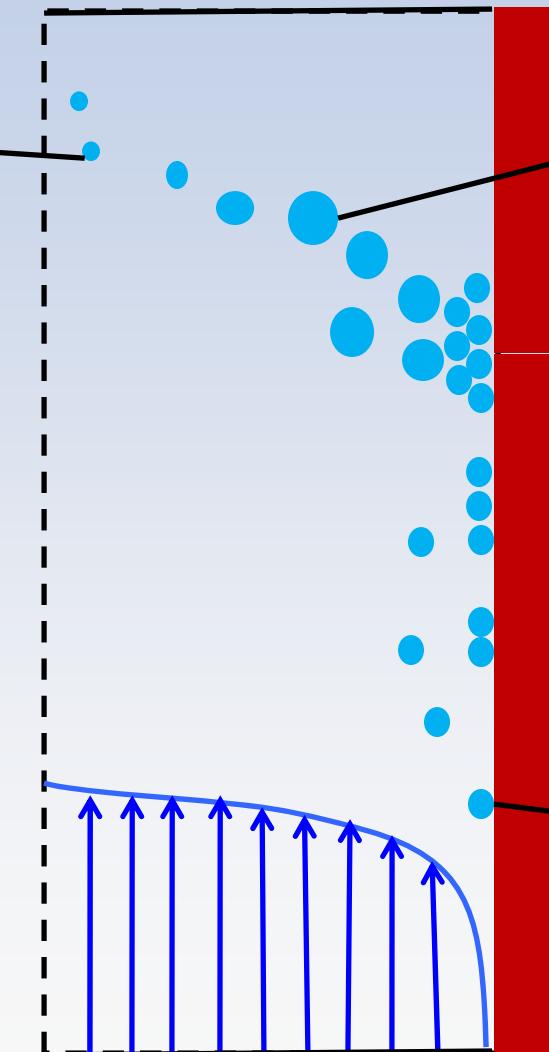
Fluid pair
variables

Coupling of RPI Wall Boiling Model with Homog./Inhomog. MUSIG



Condensation
(MUSIG
extension)

Cust.
Solv. v1



Bubble breakup
& coalescence
(MUSIG Model)

CFX 12.1

Cust.
Solv. v3

Coupling

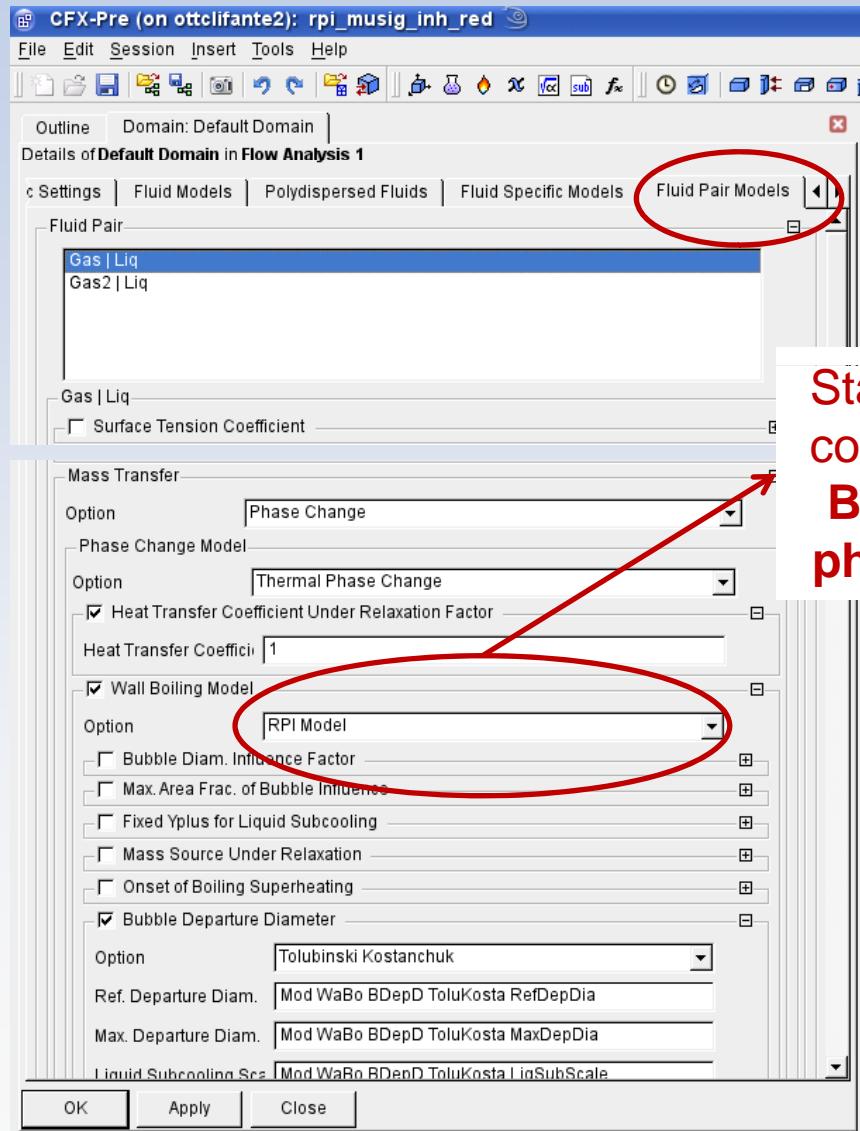
Bubble departure
Wall Boiling Model
(RPI)

CFX 12.1

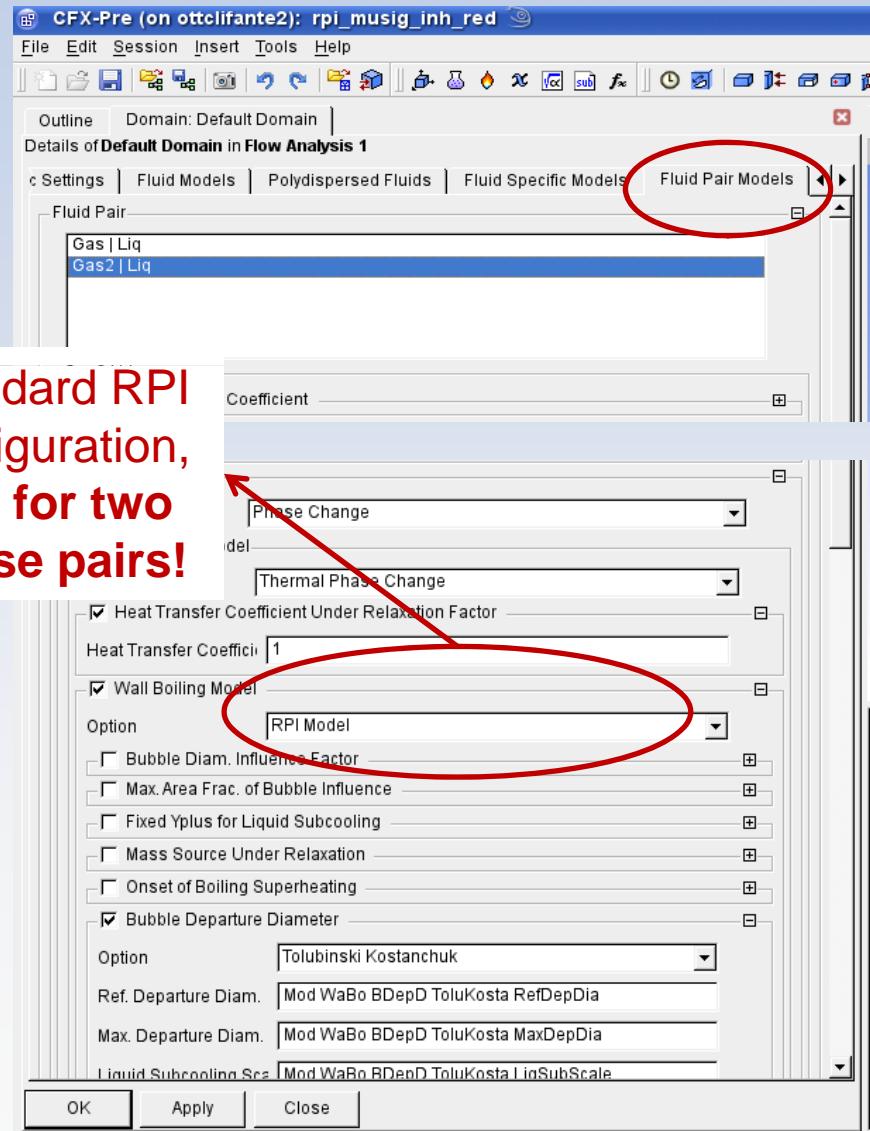
Future model extensions:

- Bubble departure diameter computed from equilibrium of forces
- Include further phenomena

CFX5Pre Customization: Inhomogeneous MUSIG & RPI



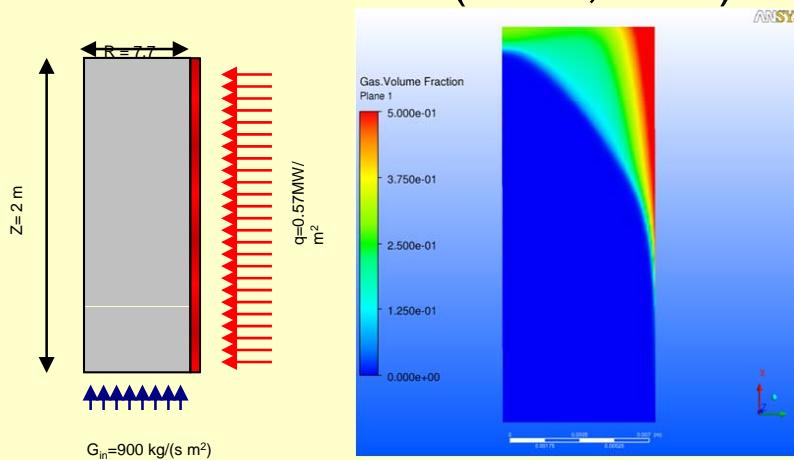
Standard RPI
configuration,
But for two
phase pairs!



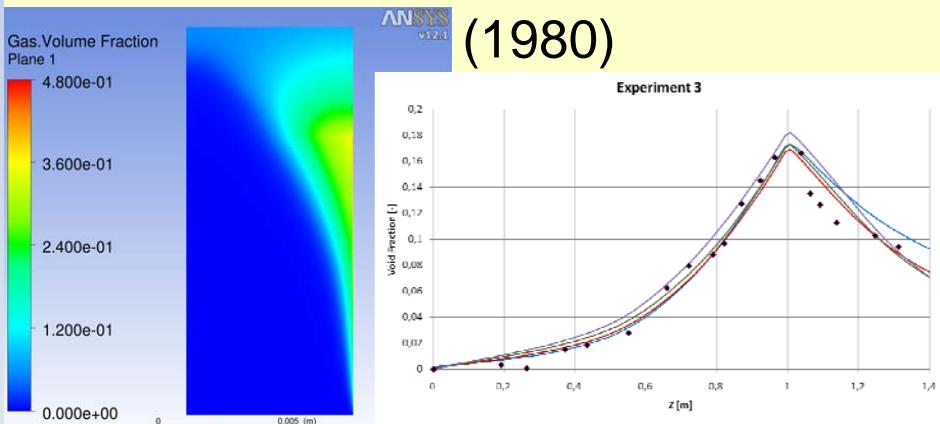
Investigated Boiling Testcases



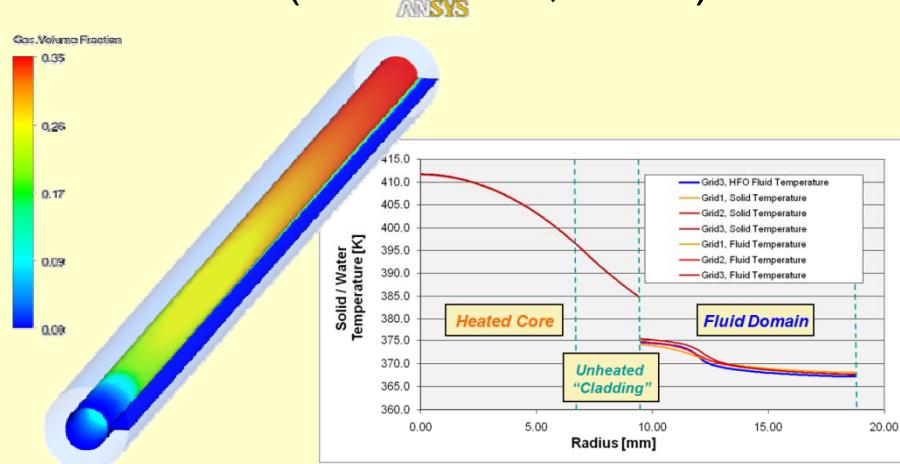
- Bartolomei et al. (1967,1982)



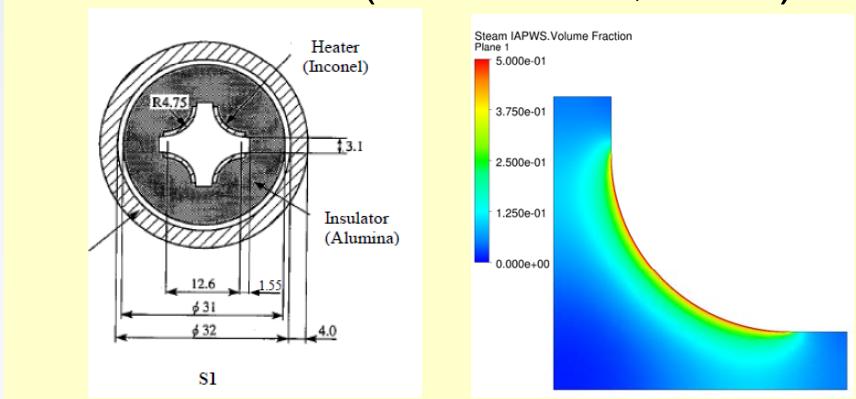
- Bartolomei with recondensation (1980)



- Lee et al. (ICON-E-16, 2008)



- OECD NEA PSBT subchannel benchmark (1987-1995, 2009)





– Model Validation –

The Bartolomej et al.
Testcase with
Recondensation
(Bartolomeij et al., 1980)



Availability of Testcases to ANSYS Customers



- ANSYS maintains a **database of validation testcases** (not only for multiphase flows)
- Bartolomei, Lee & FRIGGS testcases are available to ANSYS customers through ANSYS customer support
- Datasets of the testcases include:
 - Mesh hierarchy
 - CFD setup (baseline & parametric studies)
 - Basic post-processing and comparison to data
 - Documentation (report, paper or PPT)

Geometry & Flow Parameters

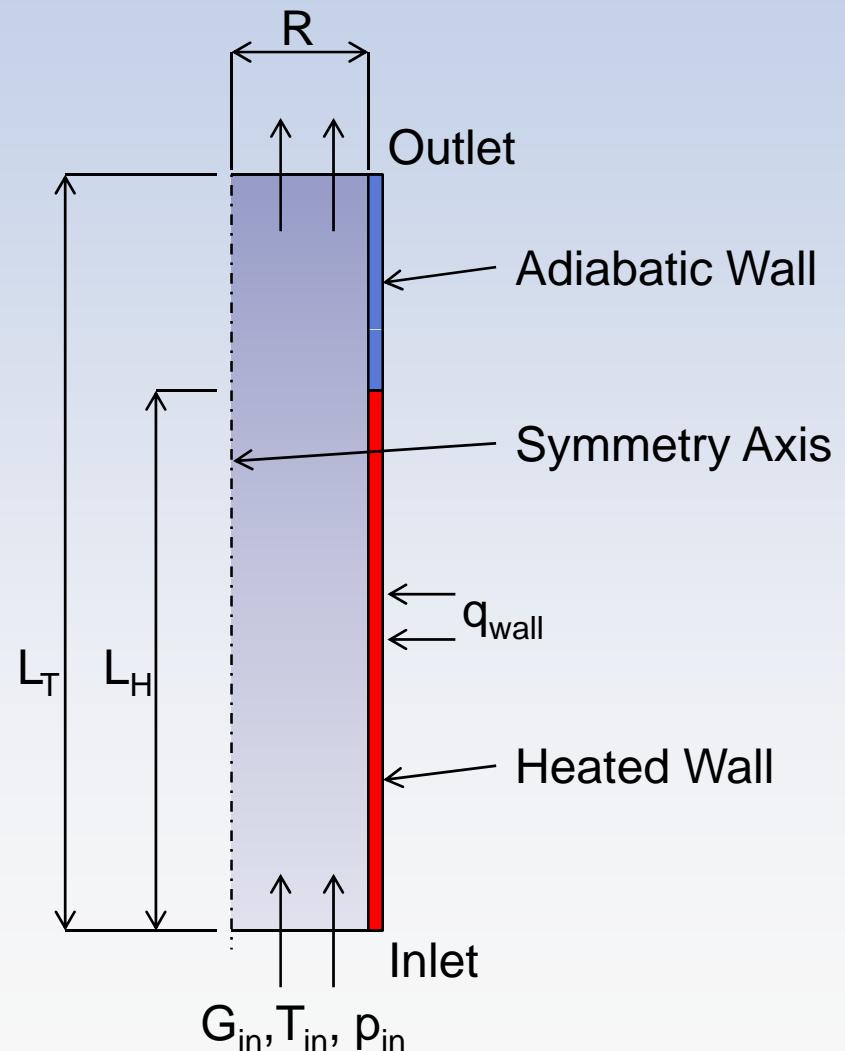


- Geometry

- Pipe flow; axial symmetry
- Inner radius of pipe $R = 6.015 \text{ mm}$
- Total pipe length $L_T = 1.4 \text{ m}$
- Heated section length $L_H = 1.0 \text{ m}$

- Flow parameters

- Upward directed water flow
- Pressure @ Inlet $p_{in} = 6.89 \text{ Mpa}$
- Parameter Investigation
 - Mass flux @ Inlet G_{in}
 - Liquid Temperature @ Inlet T_{in}
 - Wall heat flux q_{wall}



Testcase Parameters



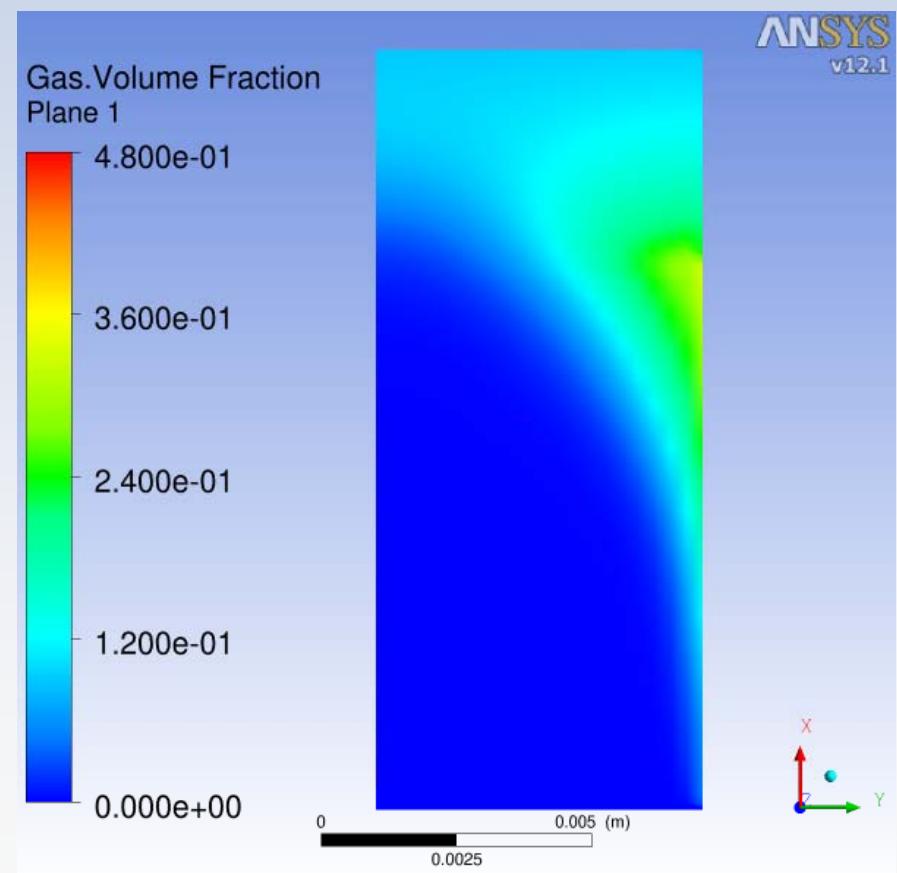
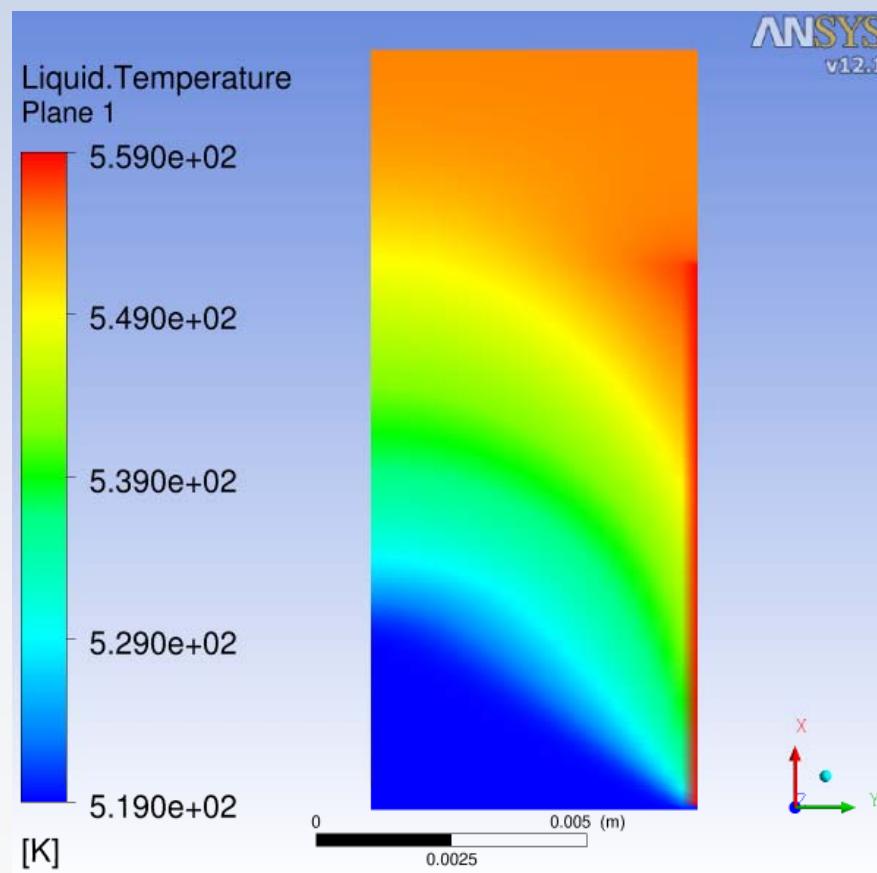
- Measurement data of zonal-averaged cross-sectional steam volume fraction distribution over pipe length are available for 3 different parameter setups:

Experiment No.	q_{wall} [MW m ⁻²]	G_{in} [kg m ⁻² s ⁻¹]	T_{in} [K]
2	1.2	1500	495
3	0.8	1500	519
5	0.8	1000	503

Experiment No. 3 (Mesh01)



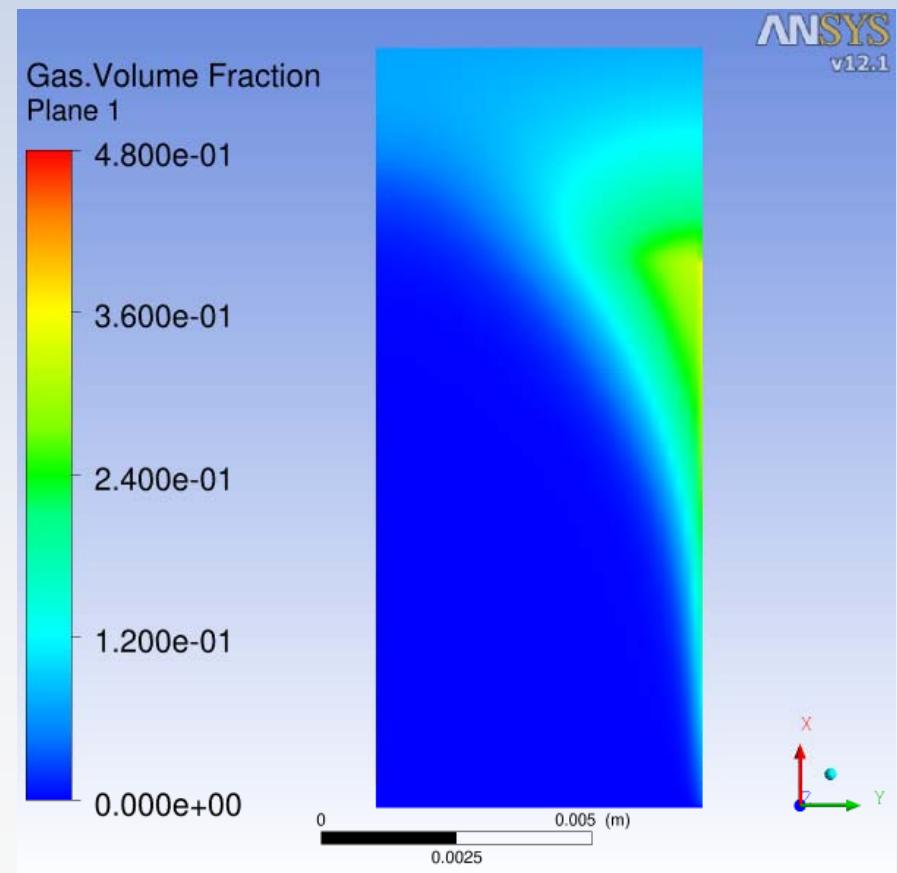
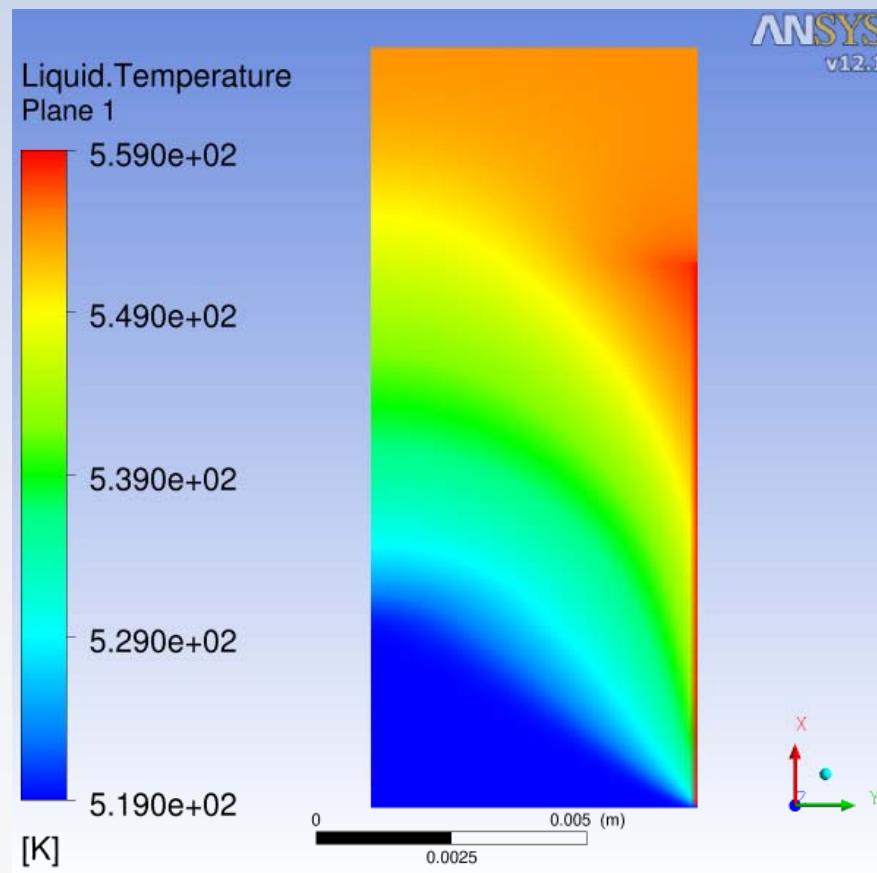
- Distribution of water temperature and steam volume fraction



Experiment No. 3 (Mesh02)



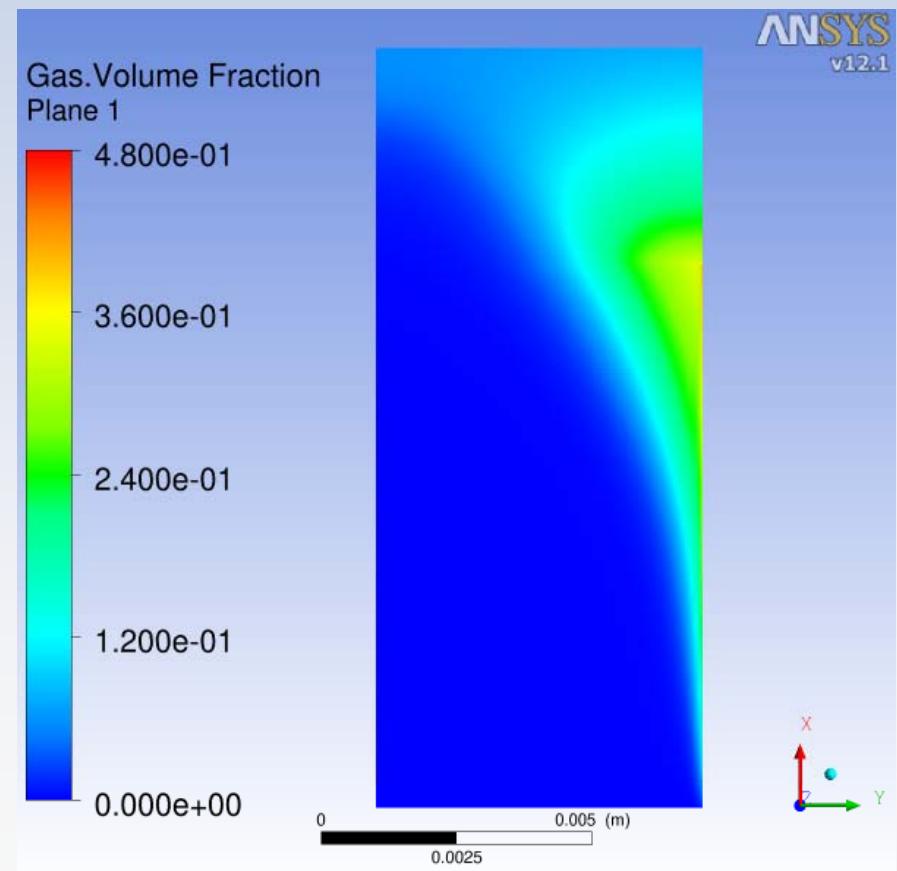
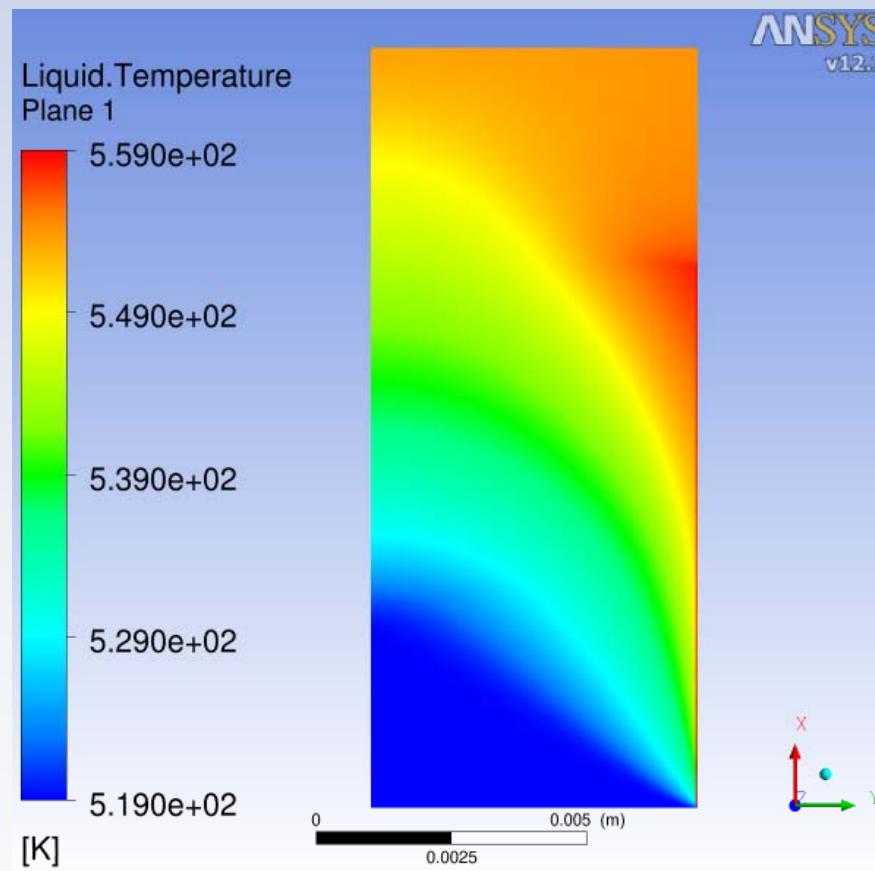
- Distribution of water temperature and steam volume fraction



Experiment No. 3 (Mesh03)



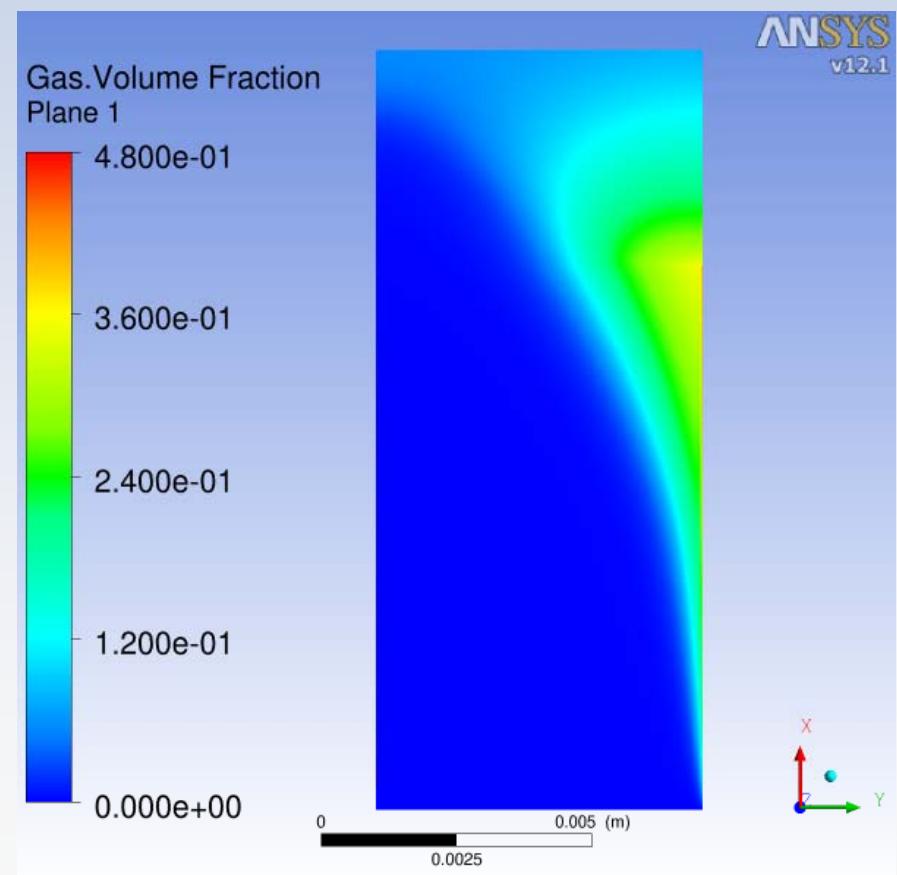
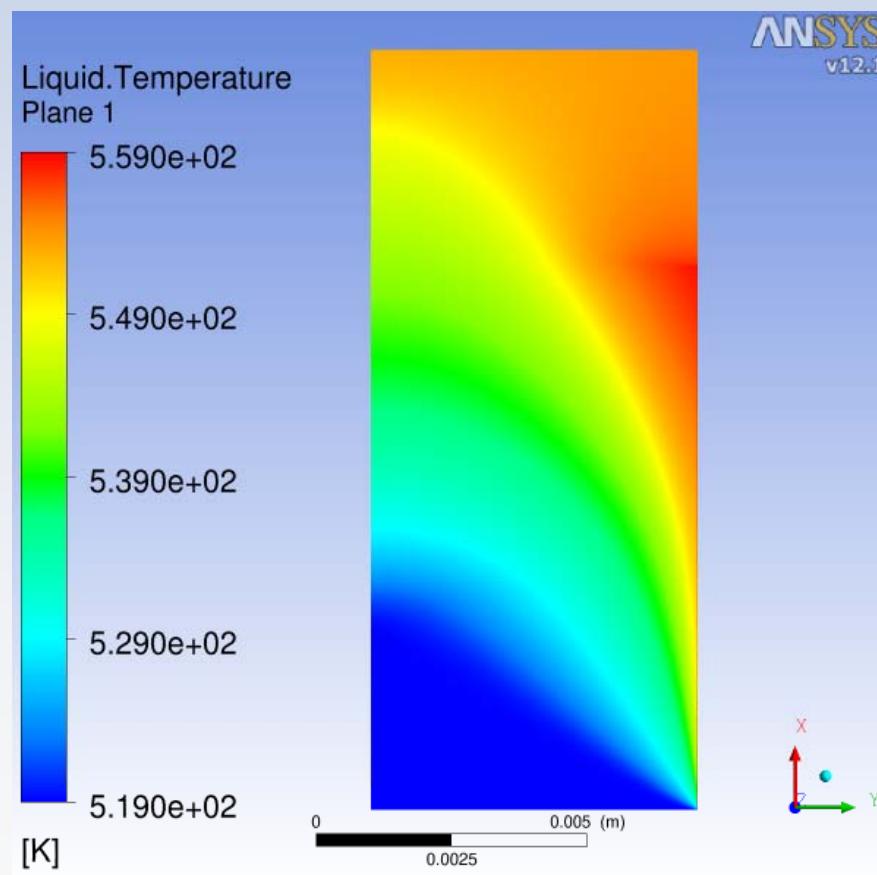
- Distribution of water temperature and steam volume fraction



Experiment No. 3 (Mesh04)



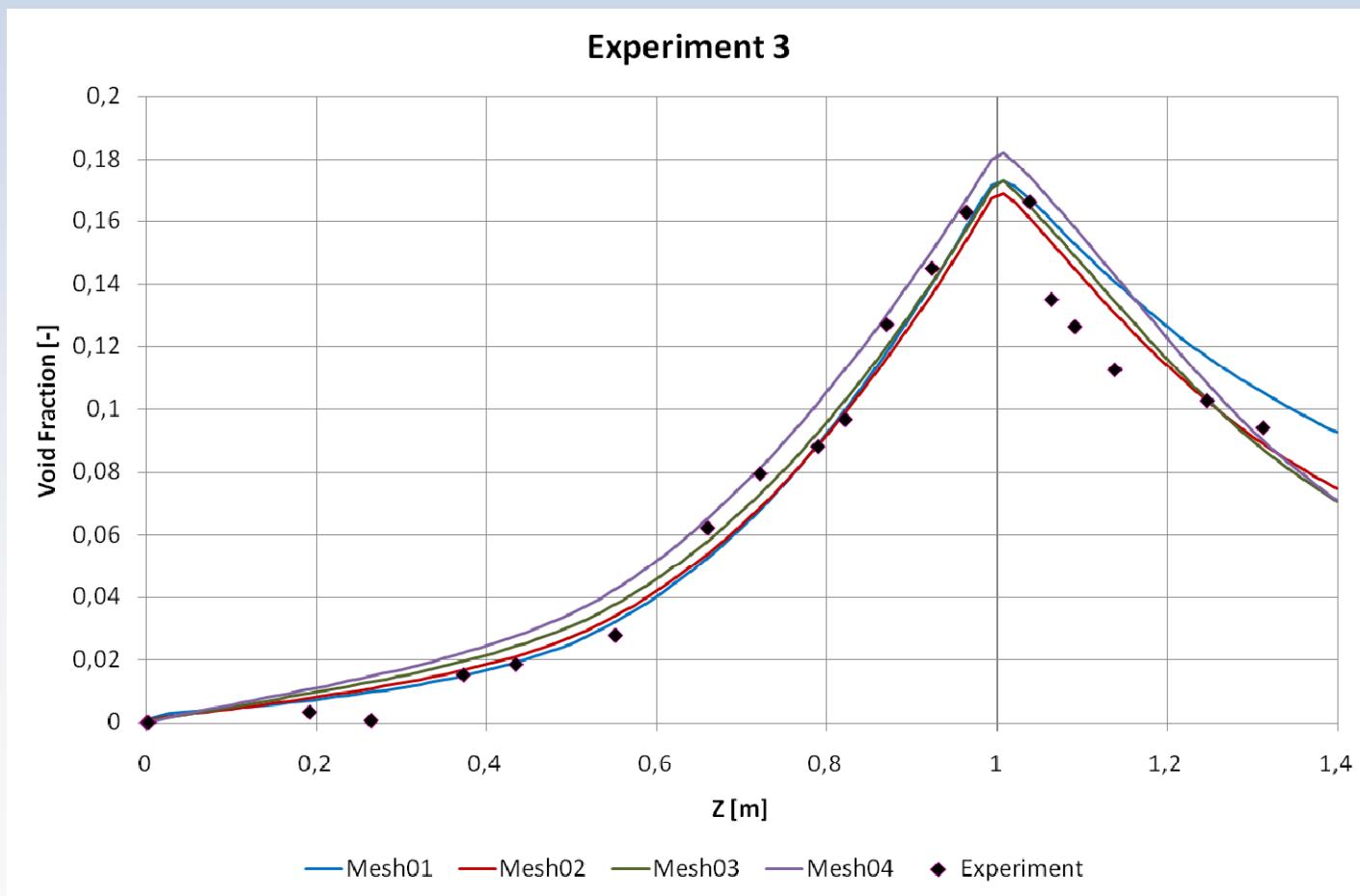
- Distribution of water temperature and steam volume fraction



Experiment No. 3

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- Comparison of cross-sectional averaged steam volume fraction to experimental data



Interface Heat Transfer Models



- Investigation of the influence of different interface heat transfer models for liquid phase
 - Ranz-Marshall (Baseline Setup)

$$Nu = 2 + 0.6 Re^{0.5} Pr^{0.3}$$

- Hughmark

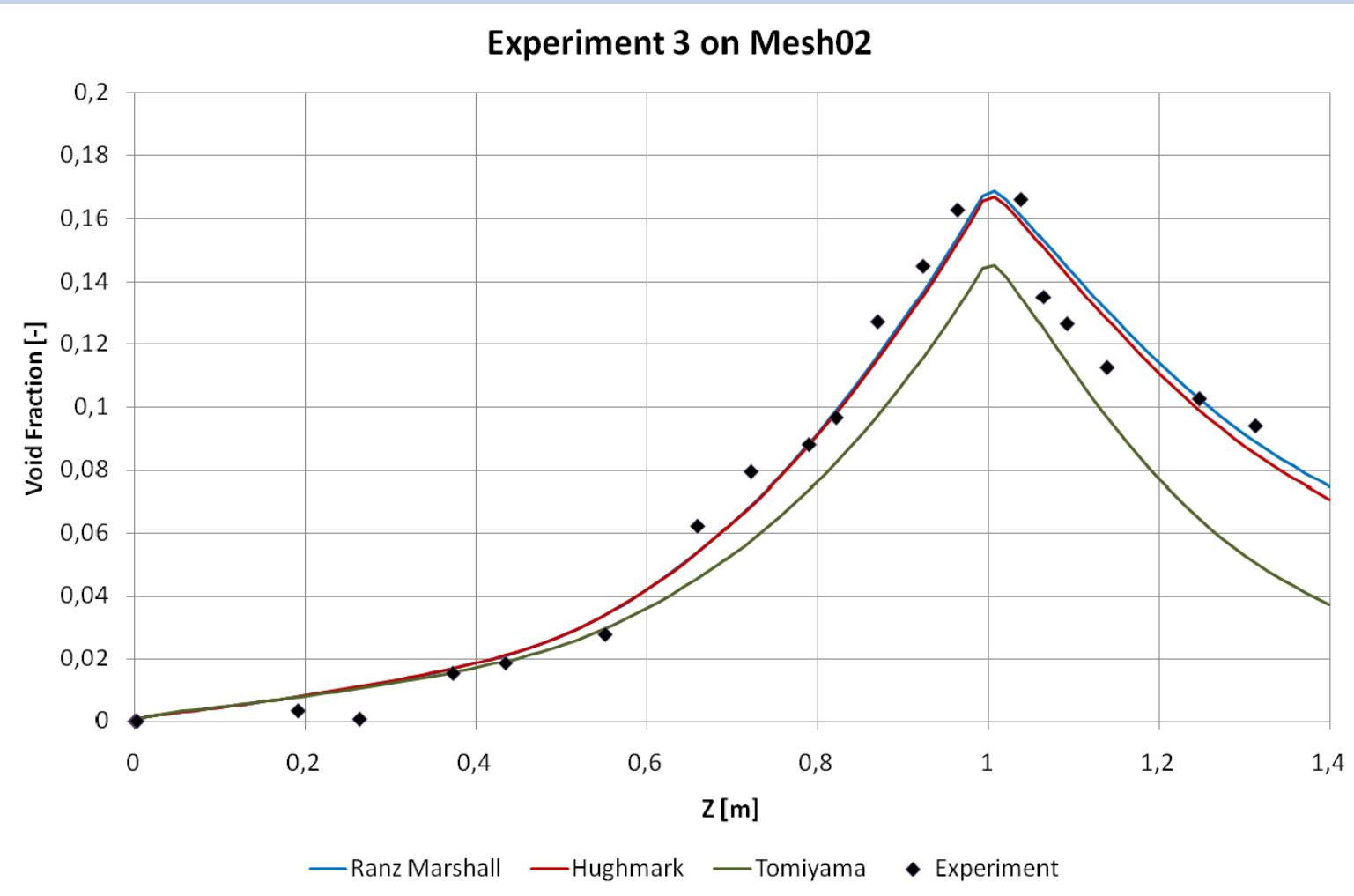
$$Nu = 2 + 0.6 Re^{0.5} Pr^{0.3} \quad 0 \leq Re \leq 776.06$$

$$Nu = 2 + 0.27 Re^{0.5} Pr^{0.3} \quad 776.06 \leq Re$$

- Tomiyama

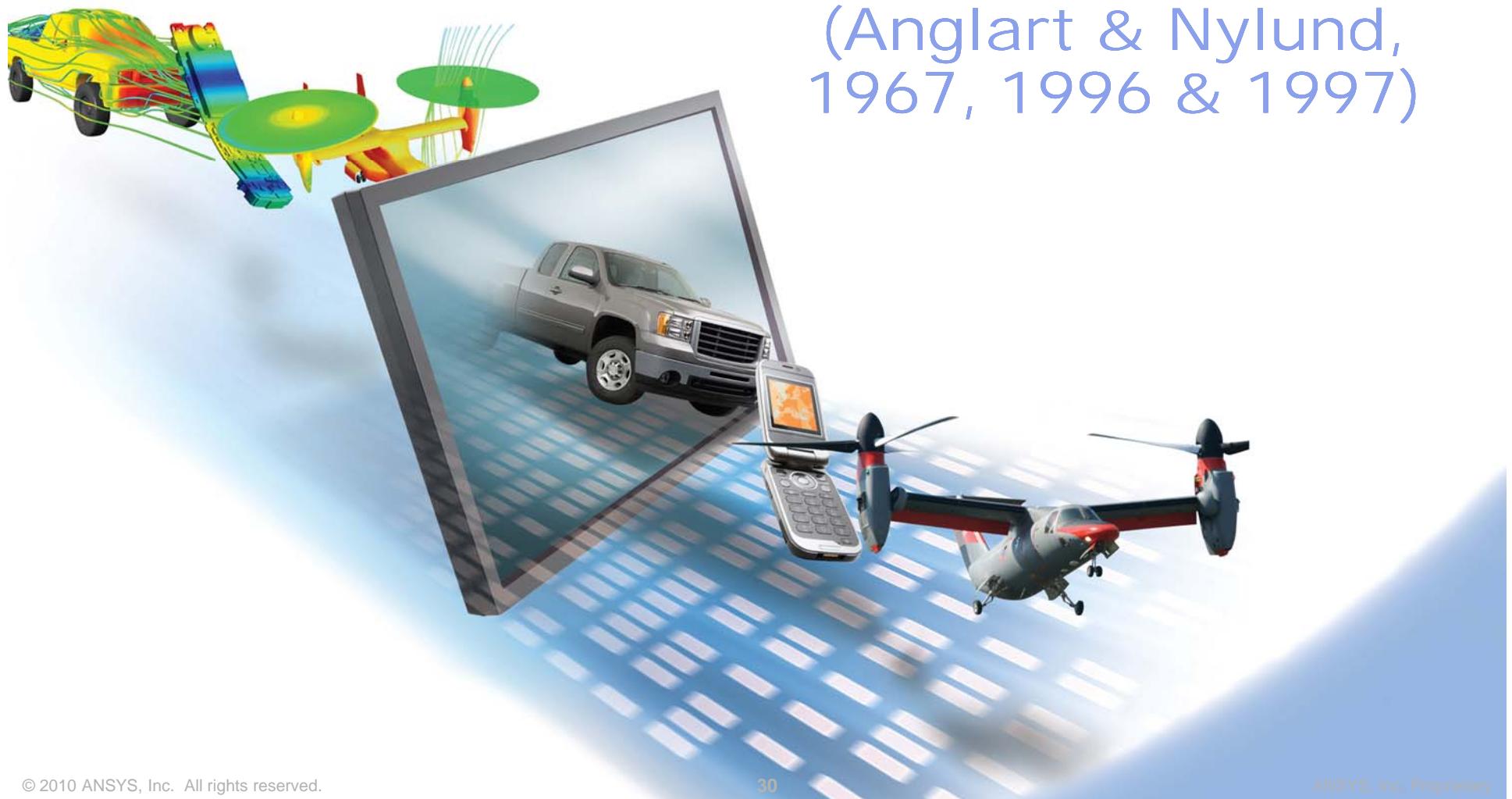
$$Nu = 2 + 0.15 Re^{0.8} Pr^{0.5}$$

Interphase Heat Transfer Model





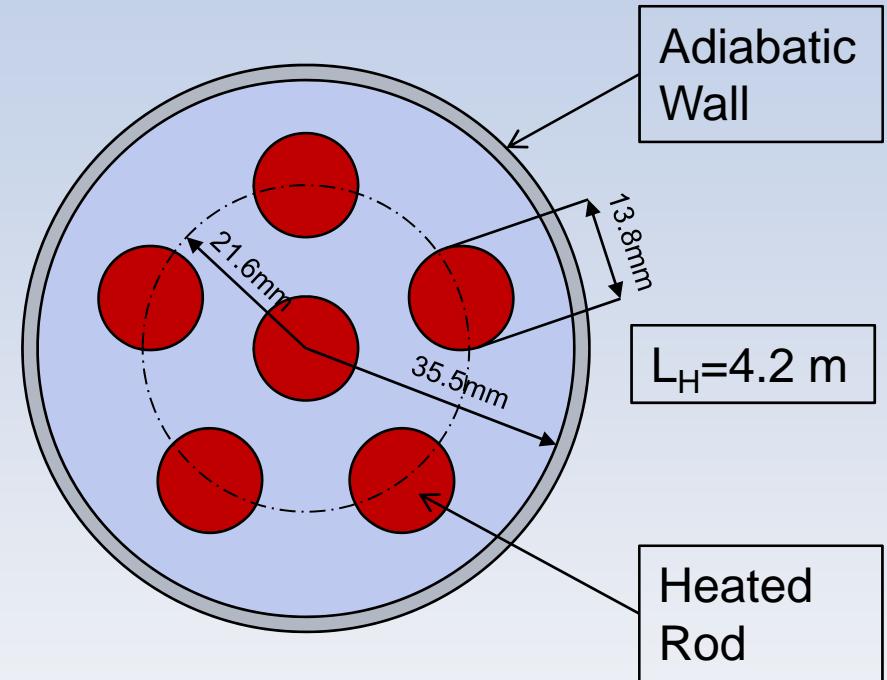
FRIGG-6a Test Case (Anglart & Nylund, 1967, 1996 & 1997)



FRIGG-6a Test Case Description



- Geometry (FT-6a)
 - Six electrically heated rods placed in a vertical adiabatic pipe
- Flow Parameters
 - Upward directed subcooled water flow
 - Mass flux @ Inlet
 $G_{in} = 1163 \text{ kg m}^{-2} \text{ s}^{-1}$
 - Pressure @ Inlet
 $p_{in} = 5 \text{ MPa}$

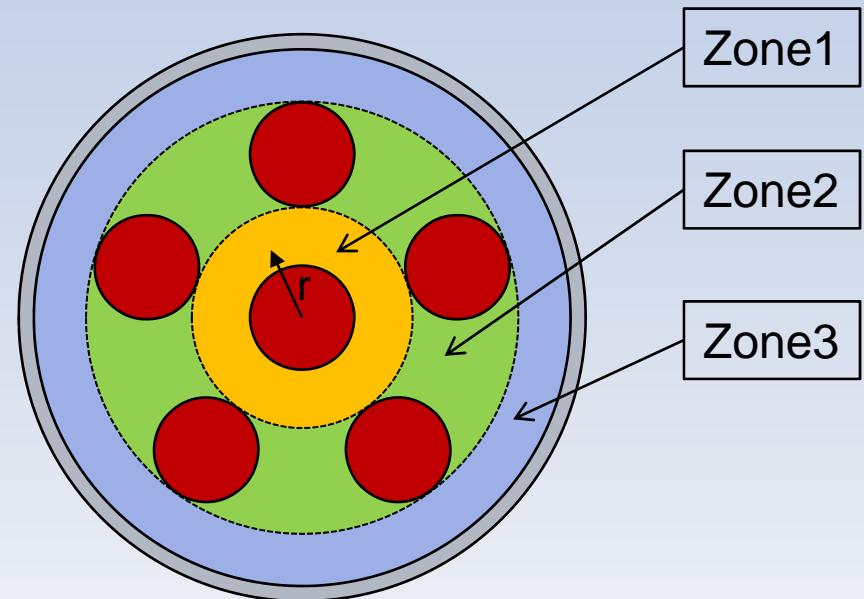


- Rod wall heat flux
 $q_{Rod} = 0.522 \text{ MW m}^{-2}$
- Liquid subcooling @ Inlet
 $T_{sub} = 4.5 \text{ K}$

FRIGG-6a Test Case Experimental Data



- Determination of experimental data by gamma ray attenuation method:
 - Measurements of area averaged gas volume fraction in different cross-sectional zones along the test section



Definition of Zones:

- Zone1 ($r < 14.6$ mm)
- Zone2 (14.6 mm $< r < 28.6$ mm)
- Zone3 ($r > 28.6$ mm)

FRIGG-6a Test Case

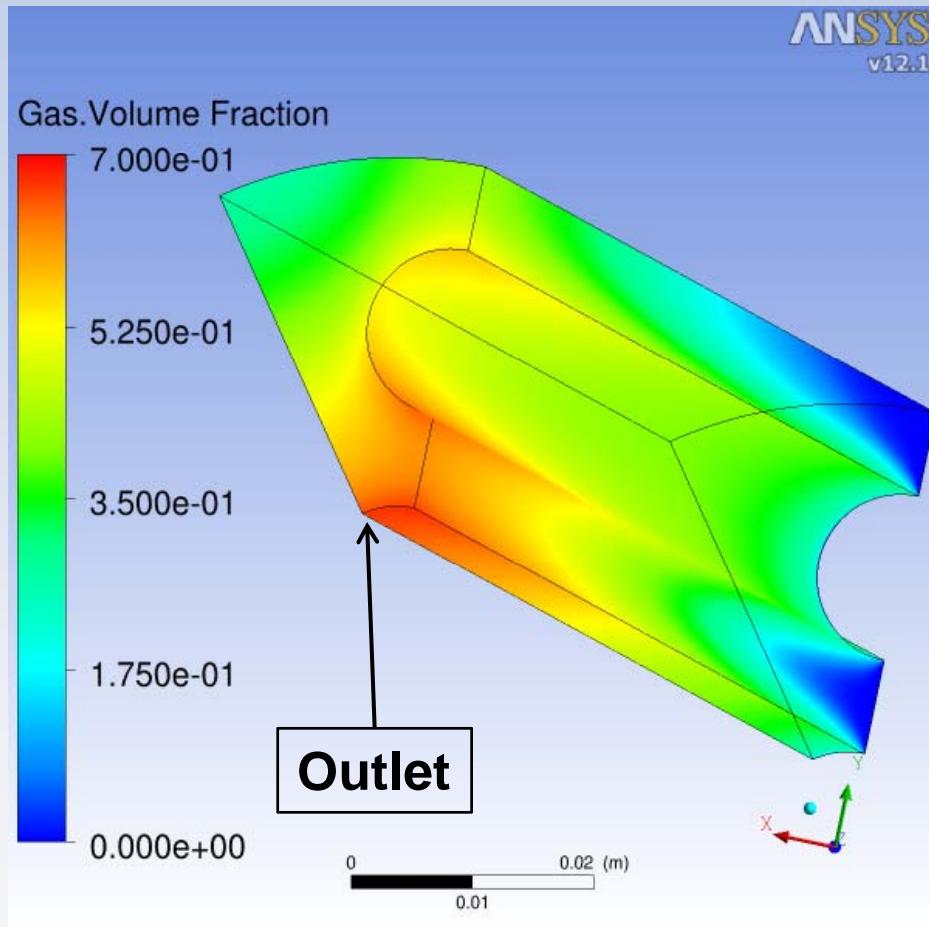
Mesh Refinement Hierarchy



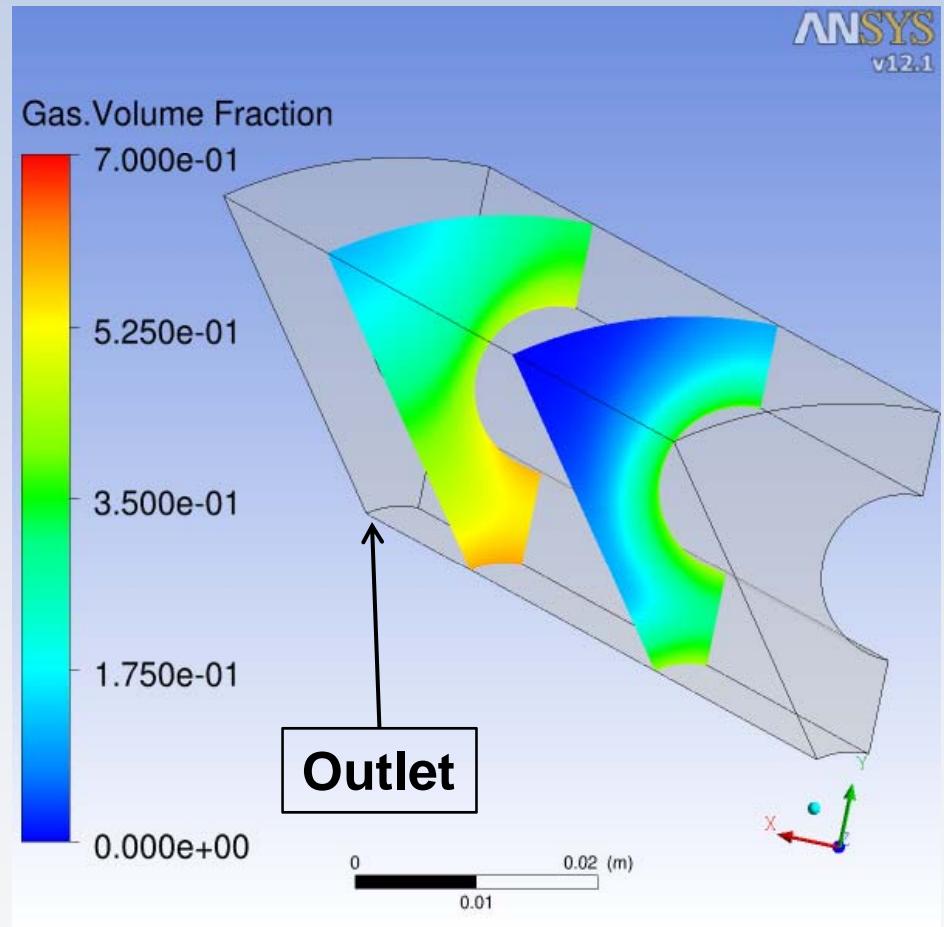
	Mesh01	Mesh02	Mesh03
No. Elements	699 x 150 (104 850)	2796 x 300 (838 800)	11184 x 600 (6 710 800)
No. Nodes	116 421	884 639	6 892 869
Max y^+	180	94	51
Min Angle [deg]	51.9	50.4	49.64
Min Determinant	0.84	0.91	0.98
Numerical Effort	~ 90 minutes @ 6 CPU's	~ 17 hours @ 16 CPU's	~ 6 days @ 40 CPU's

FRIGG-6a Test Case Baseline Setup: SST

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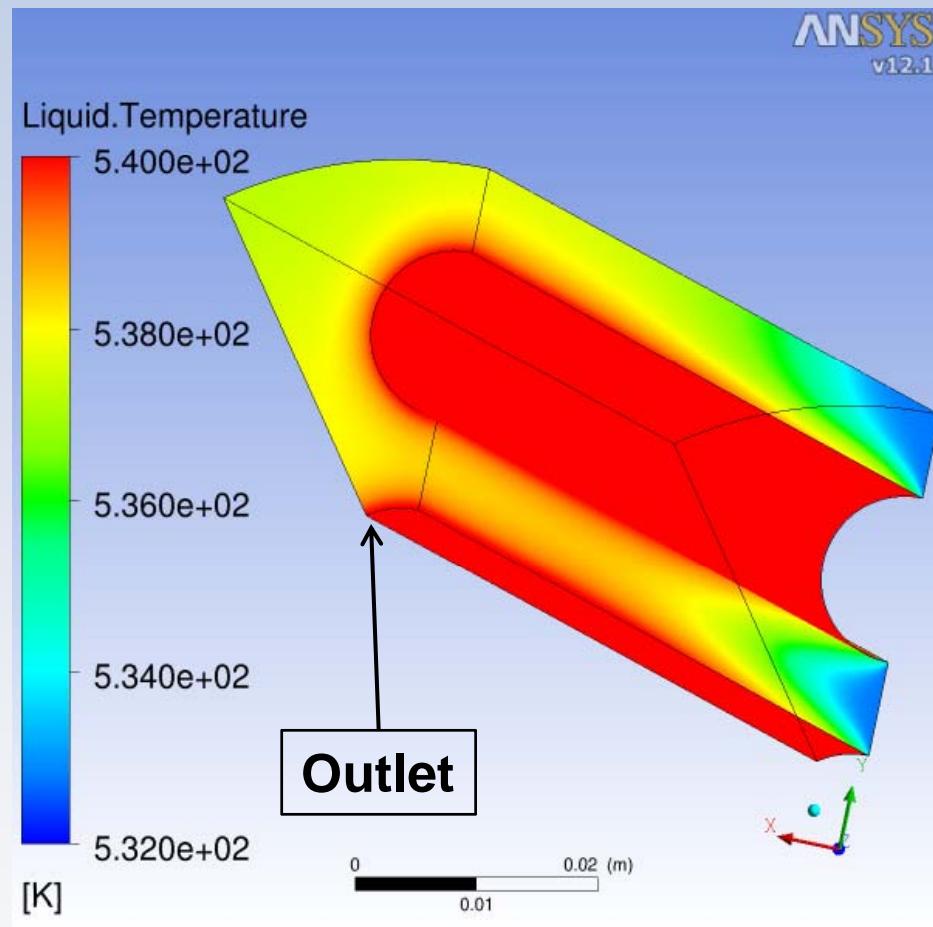
Plot of gas volume fraction (Mesh03, SST)



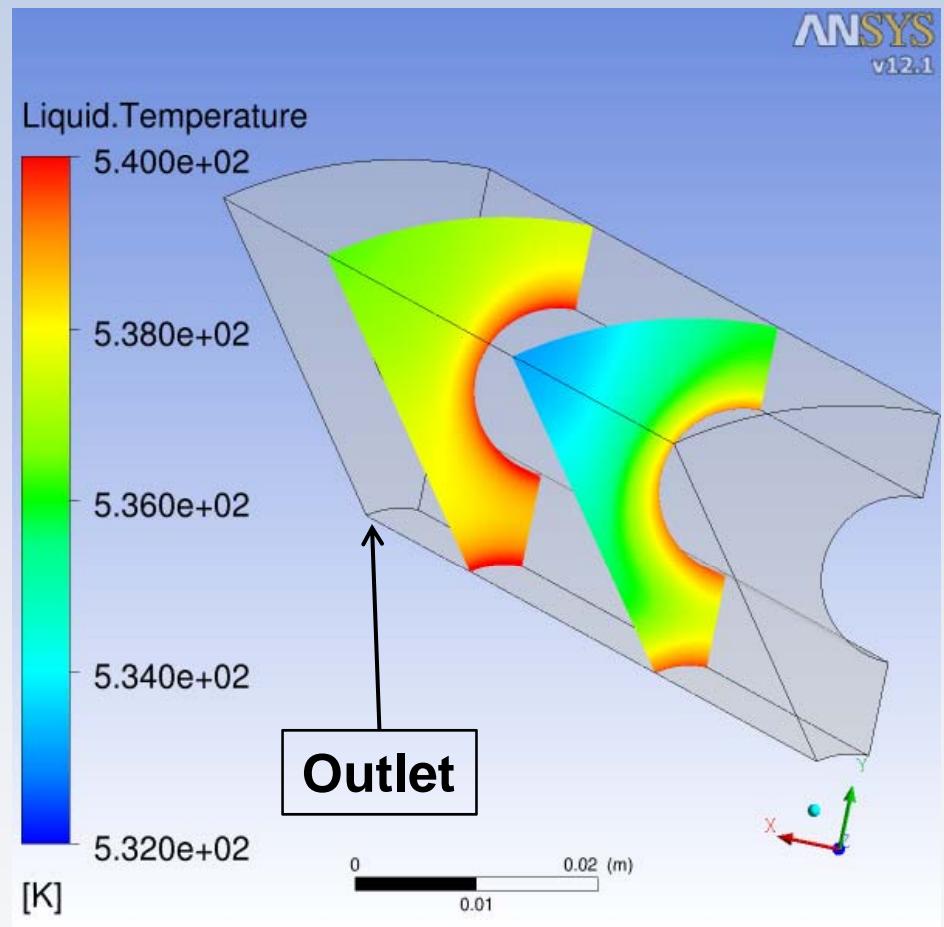
Two cross-sectional distributions of gas volume fraction (Mesh03,SST)

FRIGG-6a Test Case Baseline Setup: SST

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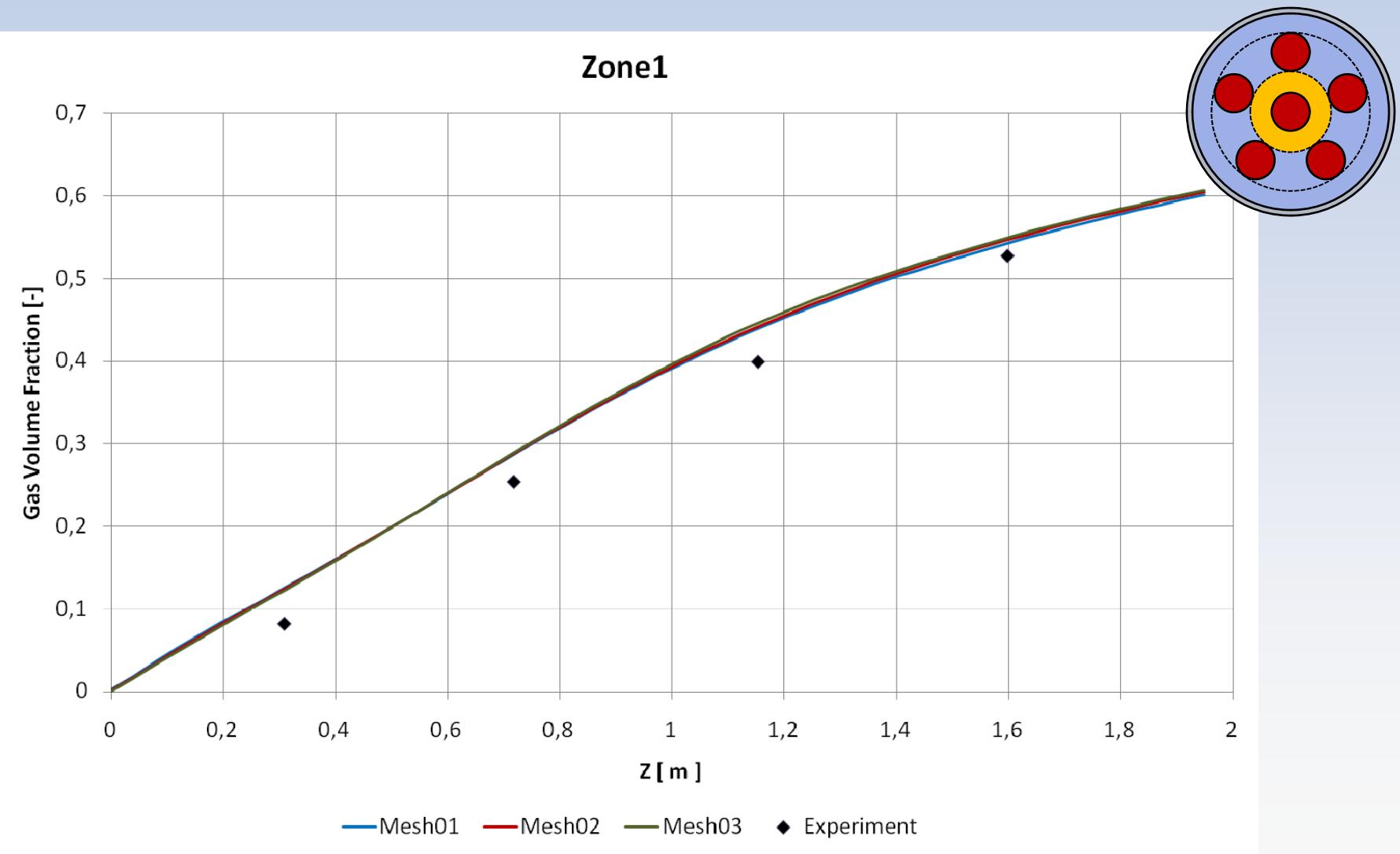
Plot of liquid temperature (Mesh03,SST)



Two cross-sectional distributions of liquid temperature (Mesh03,SST)

FRIGG-6a Test Case Mesh Comparison

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Turbulence Modeling in Rod Bundles

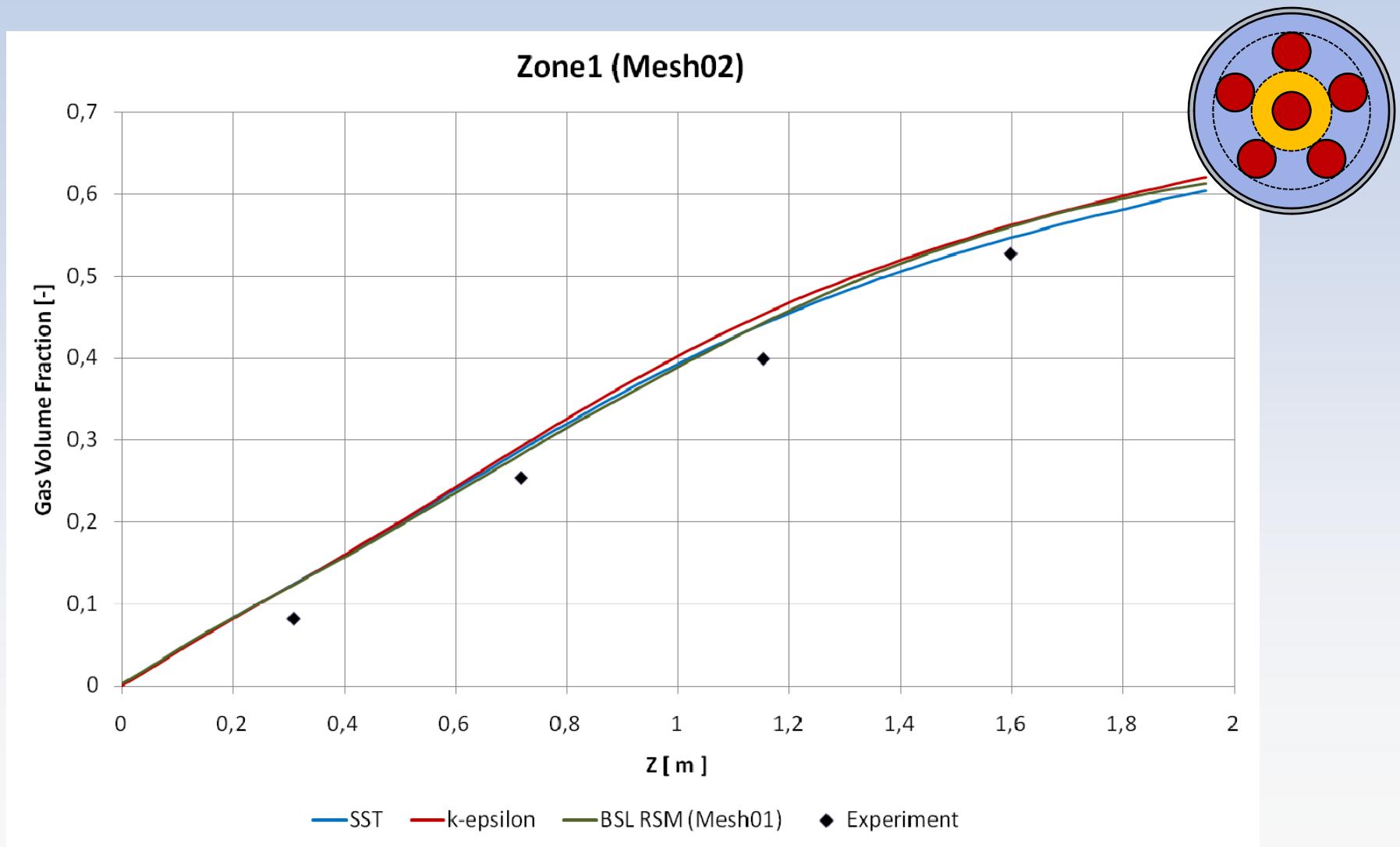


- So far good comparison, but...
 - Wall friction in rod bundles leads to secondary flows
 - Anisotropic turbulence
 - SST \Rightarrow BSL RSM
 - Does not influence so much cross-sectional averaged flow properties
 - Secondary flows affect steam & temperature distributions on wall surfaces
- Can be relevant for safety!

FRIGG-6a Test Case

Turbulence Model Comparison

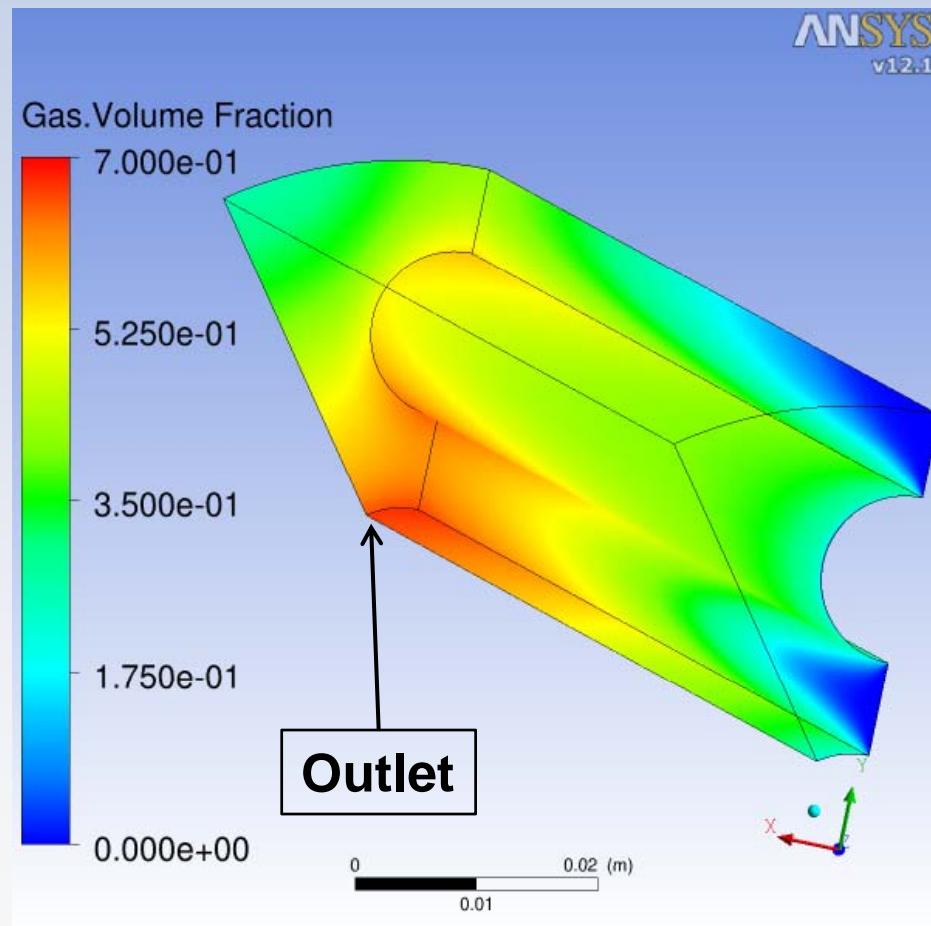
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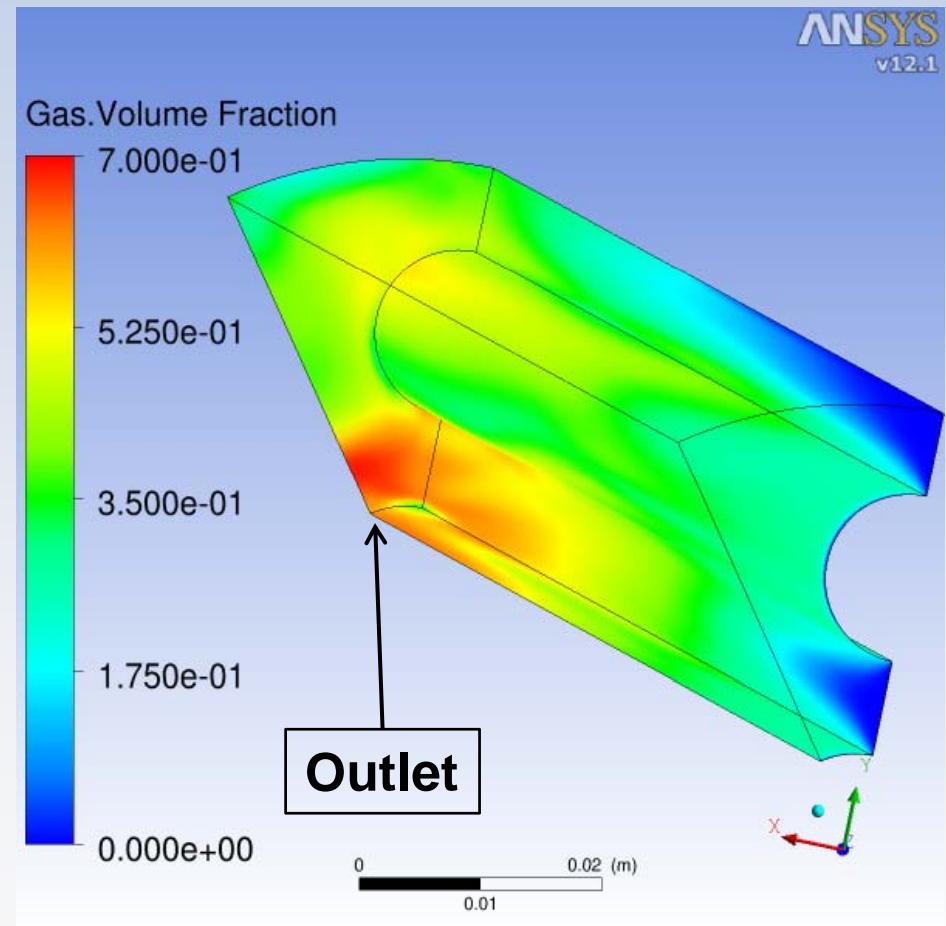
FRIGG-6a Test Case Turbulence Model Comparison



SST model



BSL RSM model

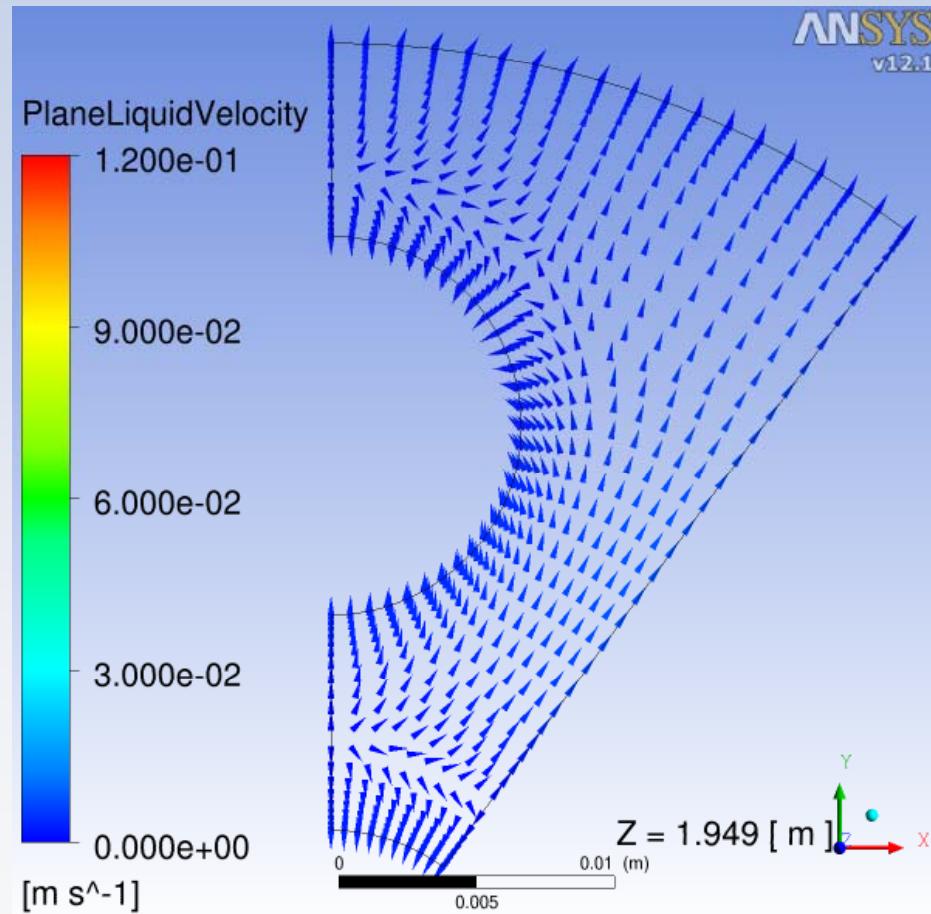


Plot of gas volume fraction

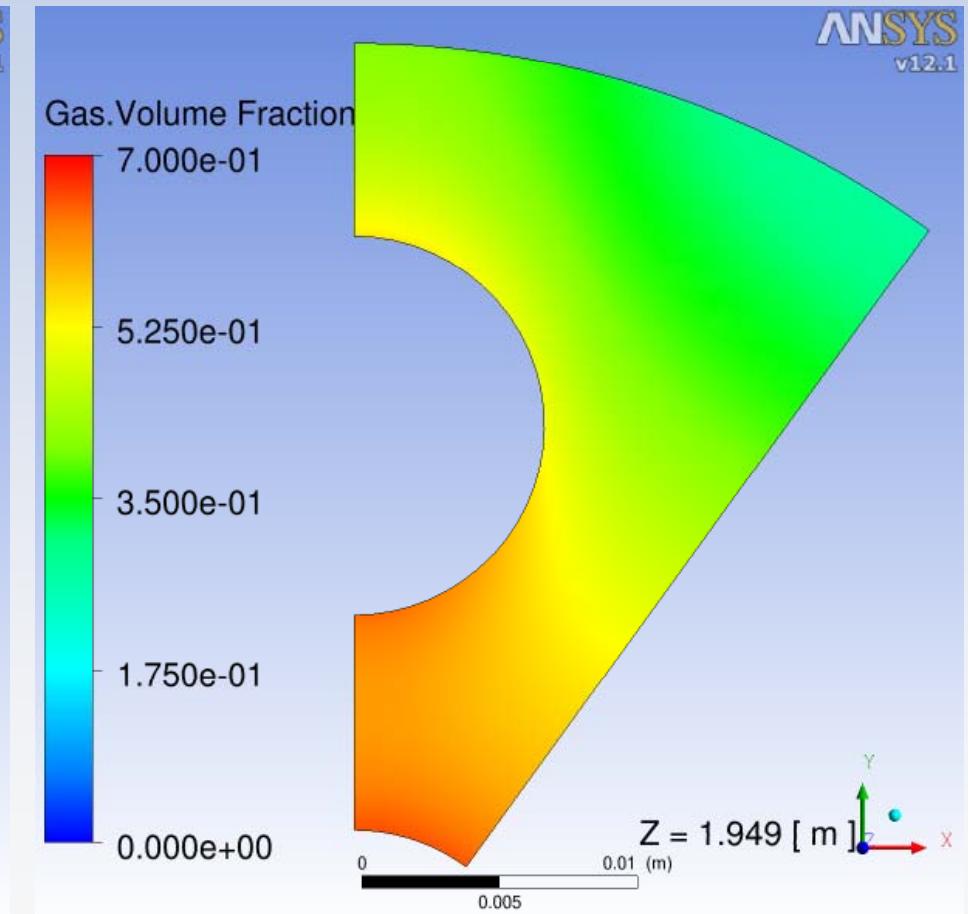
FRIGG-6a Test Case Turbulence Model Comparison

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- SST model → NO secondary flows



Plot of liquid velocity (Outlet)

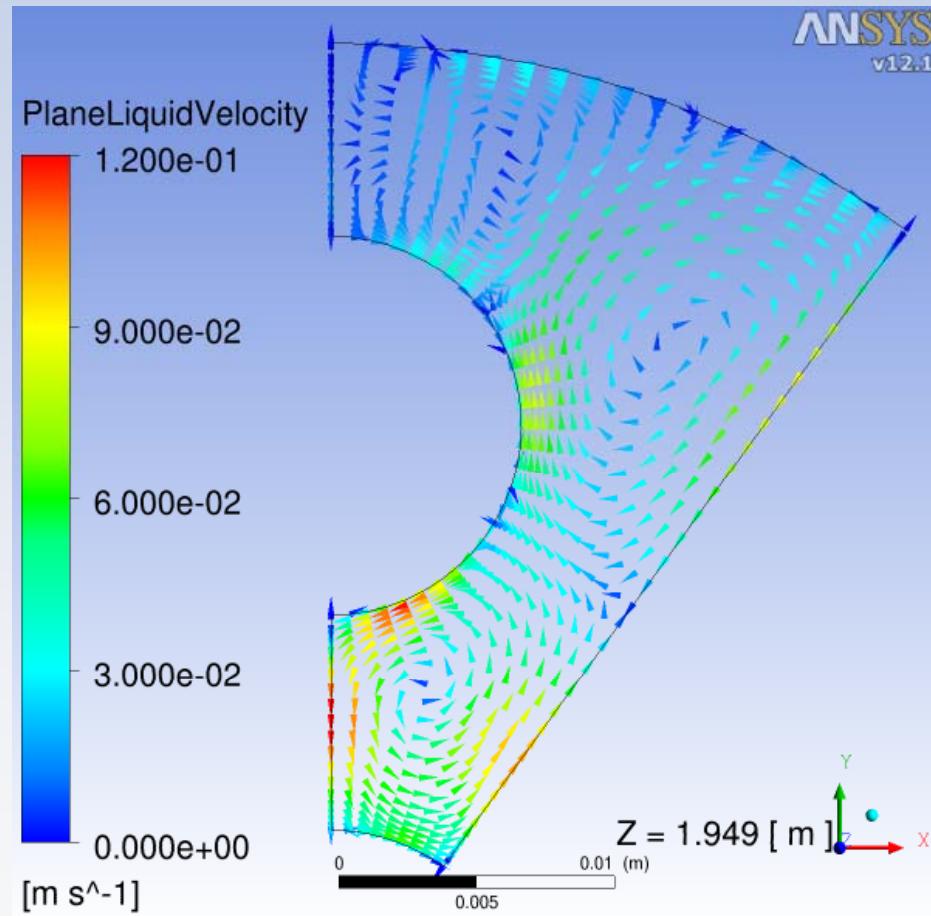


Contour plot of gas volume fraction (Outlet)

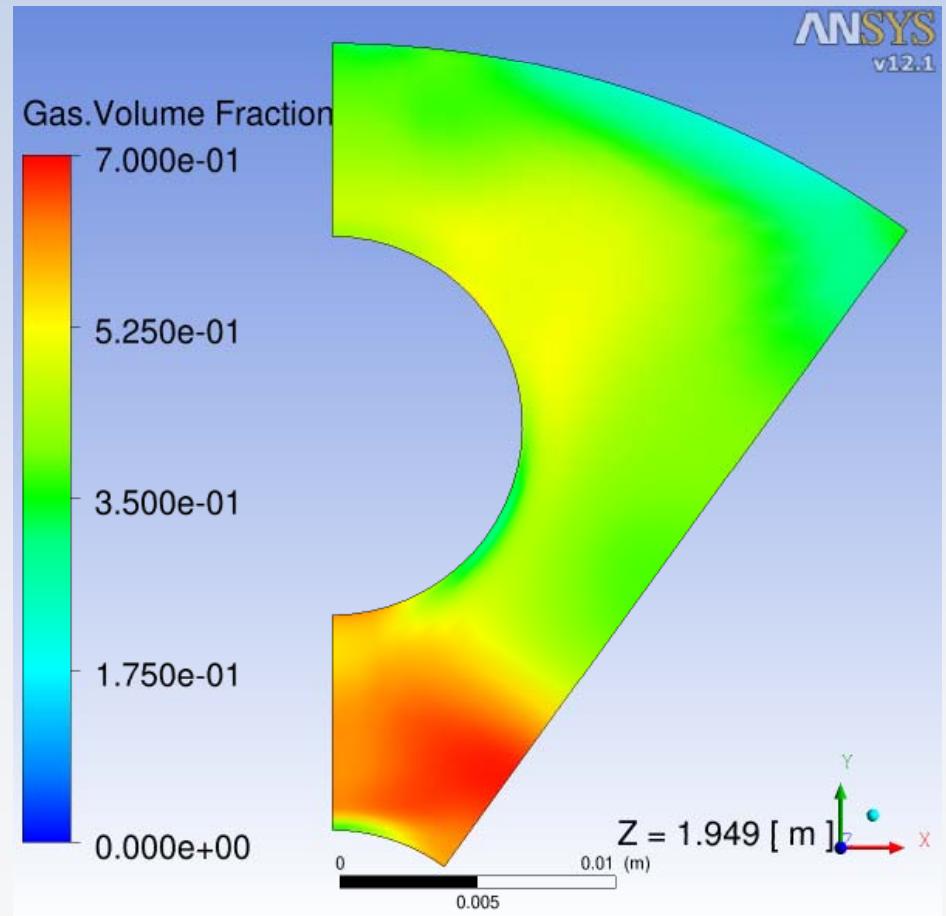
FRIGG-6a Test Case Turbulence Model Comparison



- BSL RSM model → secondary flows



Plot of gas volume fraction (Outlet)



Contour plot of gas volume fraction (Outlet)

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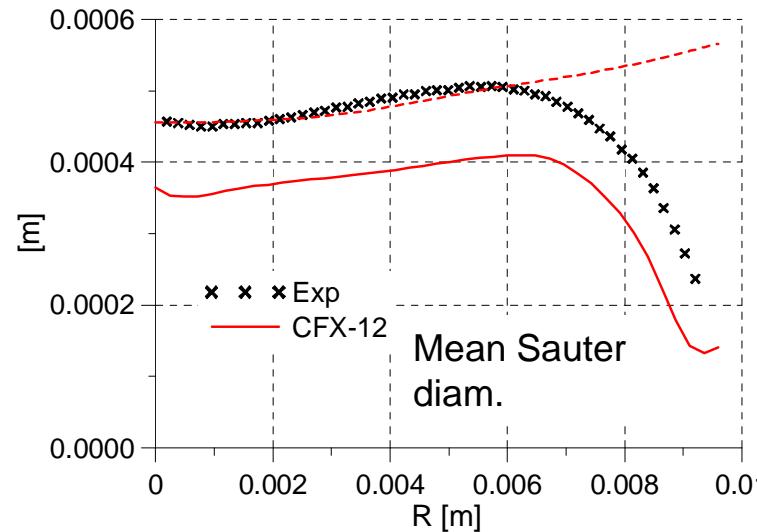
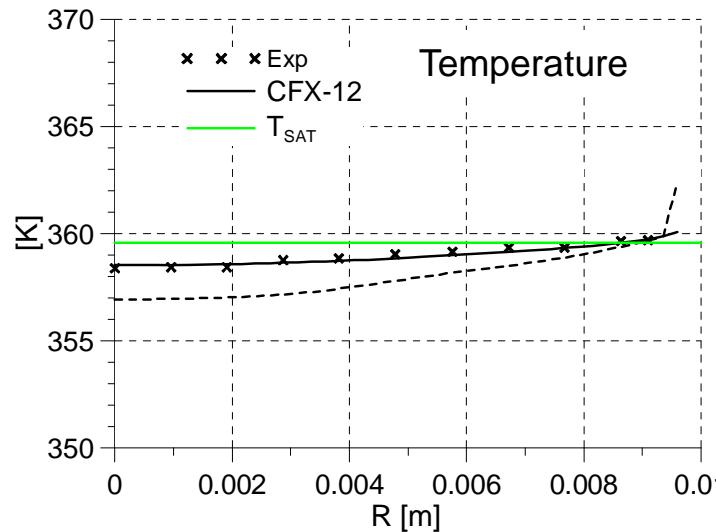
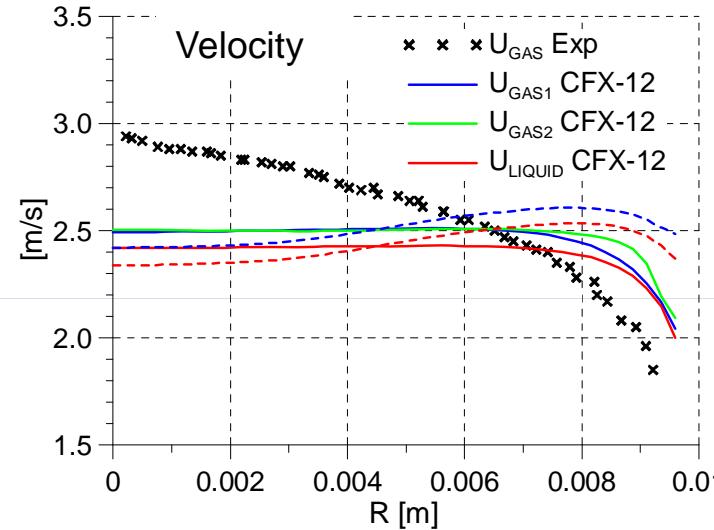
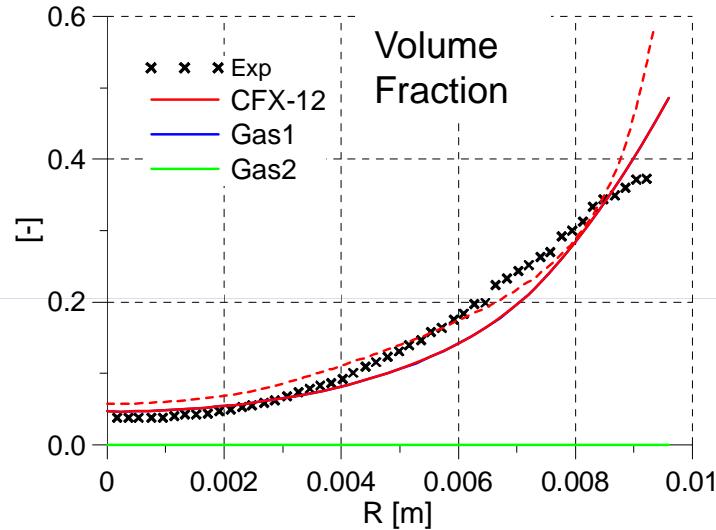
DEBORA Testcase
- RPI & MUSIG -



DEBORA Testcase: RPI & MUSIG



dashed lines – $d_B = f(T_{sat} - T_L)$; solid lines – d_B as mean Sauter diam. from MUSIG group



- Inhomog. MUSIG
- Phase change
- Breakup &
- Coalescence
- RPI

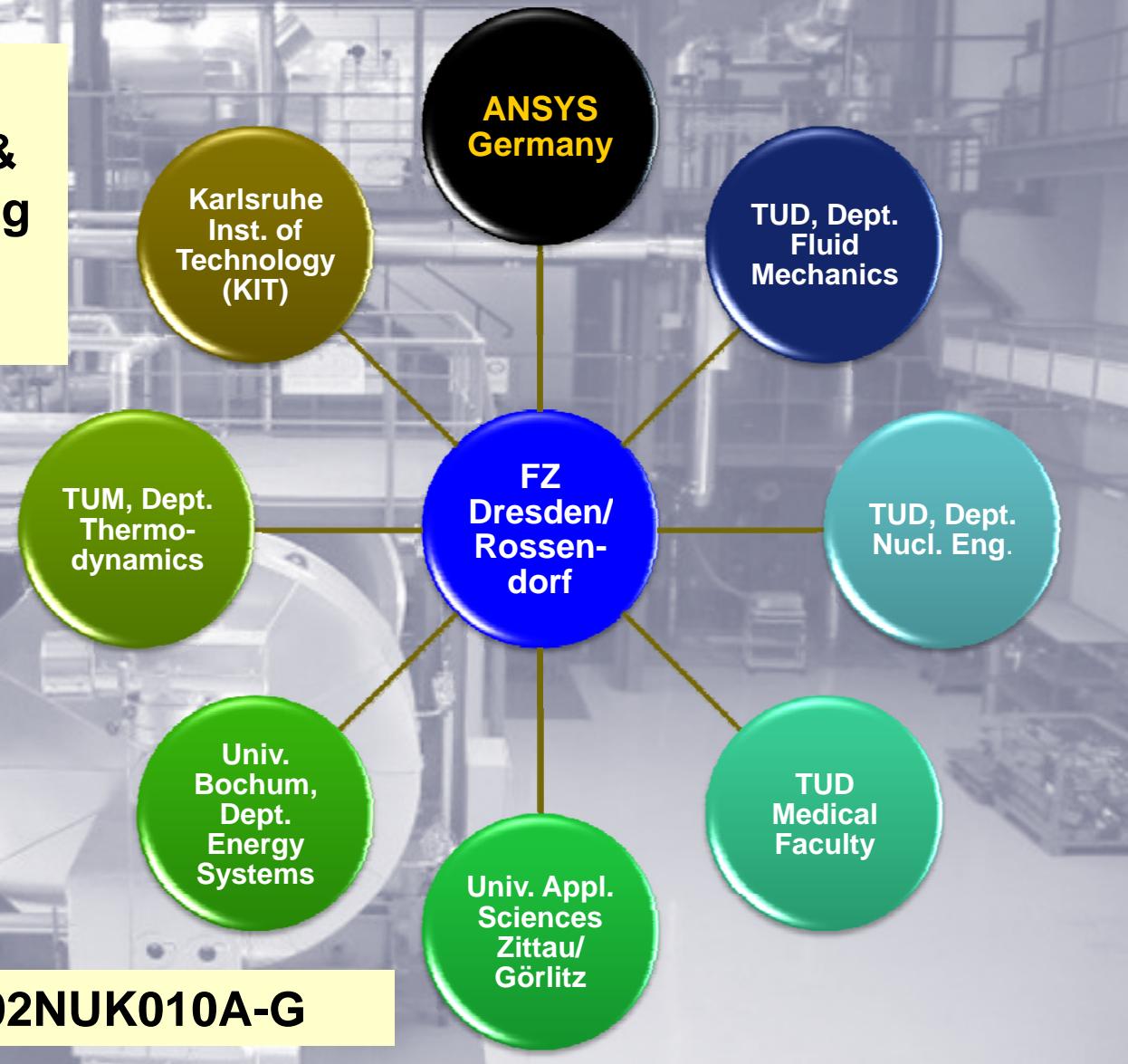
By courtesy of
E. Krepper, FZD

R&D Consortium



R&D Initiative:

**“Modeling, Simulation &
Experiments for Boiling
Processes in Fuel
Assemblies of PWR”**

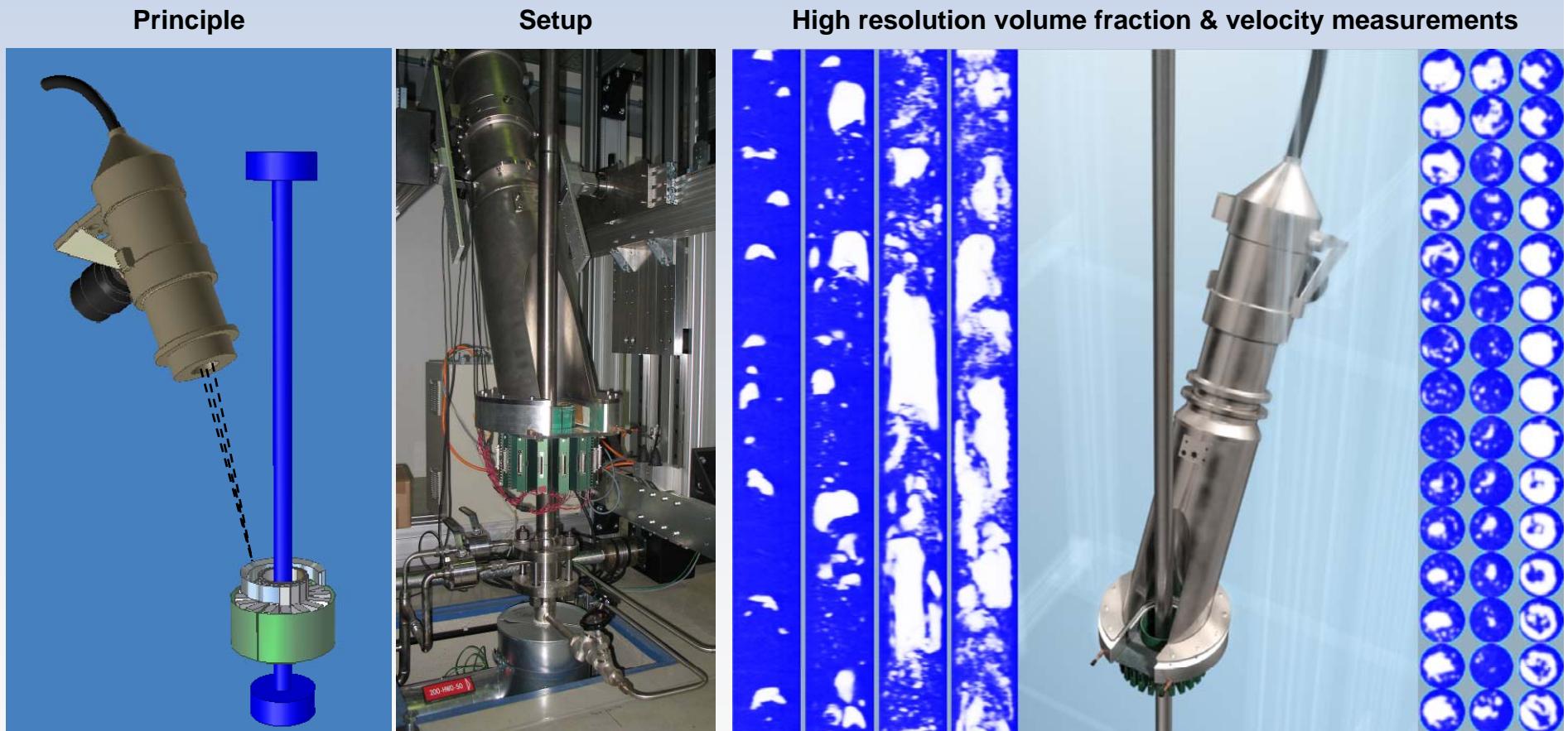


BMBF Project Grant No. 02NUK010A-G

Modeling, Simulation & Experiments for Boiling Processes in Fuel Assemblies of PWR



- Ultrafast electron beam X-ray CT (ROFEX) of heated rod bundle in titanium pipe on TOPFLOW @ FZD:



Images by courtesy of U. Hampel, F. Fischer, FZD

Summary & Outlook



- Overview on ANSYS CFD boiling model development and validation
- Continuous effort in model improvement, R&D
- Emphasis in validation on BPG, comparison to data, geometry & grid independent modeling
- Complex MPF phenomena
 - number of uncertainties remaining & requiring further investigations → detailed experiments
- Outlook:
 - Ongoing & customer driven CFD model development
 - Research cooperation with Industry & Academia
 - Extension of the wall heat partitioning in wall boiling model
 - Increase range of model applicability

Acknowledgement



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***“Modeling, Simulation and Experiments for
Boiling Processes in Fuel Assemblies of
Pressurized Water Reactors (PWR)”***

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Thank You!

