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CFD-Modeling of Turbulent Flows in a 3x3 Rod Bundle and Comparison to Experiments

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ANSYS Objective of the Investigation

 Investigation of flow conditions in HZDR ROFEX test section for boiling flow tomography measurements





Texas A&M University PIV Experiments



ANSYS Objective of the Investigation

Answering the Questions:

How the HZDR test facility can be used for CFD boiling model validation?

Where CFD results can be reasonably well compared to measurements?

→ starting with single phase



NNSYS° **Objective of the Investigation**



- Validation of single-phase flow conditions using p-cymene in a pyrex glass rod bundle
- **Comparison to PIV measurements** (Texas A&M University & HZDR)
- Isothermal, single-phase flow
- **Expected to be transient because** of previous refrigerant flow calculations







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- Thin metal sheet material & short length
 - \rightarrow initially considered as negligible \rightarrow erroneous assumption
- One spacer in measurement area
- Experimental data → spacer acts more like a flow straightener









ANSYS Computational Mesh Hierarchy Case I

	Mesh01 Coarse	Mesh02 Medium
Elements (Hex)	1.0 · 10 ⁶	$8.0 \cdot 10^{6}$

Min. Determinant	0.5
Min. Grid Angle	36
Max. Aspect Ratio	98
Max. Vol. Change	25



ANSYS Computational Mesh Hierarchy Case II

	Mesh01 Coarse		Mesh02	
			Medium	
	main	$1.0 \cdot 10^{6}$	main	$8.0 \cdot 10^{6}$
Elements (Hex)	seg1	$0.4 \cdot 10^{6}$	seg1	$3.2 \cdot 10^{6}$
	$1.4\cdot 10^{6}$		11.2 · 10 ⁶	

	main	seg1
Min. Determinant	0.5	0.6
Min. Grid Angle	34	35
Max. Aspect Ratio	120	94
Max. Vol. Change	30	10





ANSYS Computational Mesh Hierarchy Case III

Mesh02 (slice)			
Elements (Hex)	$0.6 \cdot 10^{6}$		
Min. Determinant	0.7		
Min. Grid Angle	38		
Max. Aspect Ratio	1		
Max. Vol. Change	5		





ANSYS Computational Mesh Hierarchy Case IV

	Mesh01 Coarse		Mesh02 Medium	
	main	$0.6\cdot 10^6$	main	$4.8 \cdot 10^{6}$
Elements (Hex)	seg1	$0.4 \cdot 10^{6}$	seg1	$3.2 \cdot 10^{6}$
	1.0 · 10 ⁶		8 · 10 ⁶	

	main	seg1
Min. Determinant	0.6	0.6
Min. Grid Angle	36	35
Max. Aspect Ratio	120	94
Max. Vol. Change	7	10





Mesh02

ANSYS Computational Mesh - Mesh02



ANSYS Computational Mesh – Case II and IV



ANSYS Boiling Model Demands



- Domain is subdivided in three parts for further investigations considering heat
- Constant cell thickness around rods





Boundary Conditions & Material Properties



- Normal Speed u₁, u₂, u₃ Turbulence Intensity 5%
- Domain: Reference Pressure 1 bar Isothermal 28 °C Turbulence: SST, BSL RSM
- Walls: No Slip Wall, Smooth Wall for pipes, rods & dividers
- Outlet: Average Static Pressure Relative Pressure 0 bar

Material: P-cymene Dynamic Viscosity Density

0.761·10⁻³ kg m⁻¹ s⁻¹ 850.79 kg m⁻³



- Imbalances
- Residuals (MAX, Localization)
- Stability of Monitored Quantities

< 1%

Max. Res. Criterion 10⁻⁴ / 10⁻²

U, V, W, P (inst. / transient avg.)



Residuals above 10⁻⁴ and also above 10⁻³ concentrated in outlet vessel



Qualitative Analysis: Flow Field Characteristics













ANSYS Secondary Flow **SST Mesh Refinement**



• similar to Steady State Results, SST

ANSYS Mirrored Case Appearance

Comparing Average Velocity Profiles Plane 2 and 4 Different Convergence Criteria, Velocity Range [1.0; 1.8]



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ANSYS Mirrored Case Appearance

Comparing Average Velocity Profiles Plane 2 and 4 switched in the 10⁻⁴ case



→ Mirrored case established

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averaging started at 1.6s, u₃@Mesh01



Quantitative Analysis: Comparison to Experimental Data



Location of profile lines for data comparison



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ANSYS Experimental Comparison

Symbols



Legend













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Plane 4, Mesh01, transient









Grid Spacer Influence

Plane 2

Plane 4



 $\mathbf{u_1}$



Turbulence Model Comparison



SST vs. BSL RSM





Turbulence Model Comparison for Fully Developed Flow



Plane 4

SST vs. BSL RSM

Plane 2



Scaling of CFD results based on comparison of fully developed flows:

 $v_{CFD,scaled} = f \cdot v_{CFD}$ with $f = \frac{m_{EXP}}{m_{CFD}} \approx \frac{v_{max,subchannel,EXP}}{v_{max,subchannel,BSLRSM}} = \frac{0.73}{0.79} = 0.924$

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- Flow in inlet plenum very unsteady, causes asymmetry and swirl, inflow conditions not well controlled in experiments
- Non negligible impact of the spacer grid
 - → swirl reduction like for a flow straightener despite the short length and thin sheet metal material
- Uncertainties in experiment and measurements:
 - Flow inlet into inlet plenum with riffle hose of unclear position
 - Mass flow rates not accurately specified or controlled
 - Distance between camera position and measurement plane not accurately measured → "scaling effect" on velocities
- Geometry models $I \rightarrow IV$: from qualitative to quantitative agreement
- Finally the Case IV results are grid independent and show reasonably good comparison to experiments, at least for higher elevation
- For boiling flow experiments more effort for reliable inflow conditions to the test section is required
 - \rightarrow Otherwise the use for CFD model validation would be questionable



Acknowledgements

SPONSORED BY THE

Federal Ministry of Education and Research This research has been supported by the German Ministry of Education and Research (BMBF, Grant No. 02NUK010G) in the framework of the R&D funding concept of BMBF "Basic Research Energy 2020+", the German CFD Network on Nuclear Reactor Safety Research and the Alliance for Competence in Nuclear Technology, Germany.



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Questions ?



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